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COMPUTER PROGRAM FOR PREDICTION
OF AXIAL FLOW TURBINE PERFORMANCE

by

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ABSTRACT:

The report presents a computer program for prediction of performance of single-stage axial turbines of given geometry. The three-dimensional method developed by Vavra is applied, taking account of streamline curvatures and slopes, as well as enthalpy and entropy gradients in the solutions of the equation of motion, and of boundary layer thicknesses in the continuity equation.

A choice among five different loss correlation methods and two flow angles correlations is offered. Loss coefficients and flow angles are automatically calculated from blading geometry and actual flow conditions for every streamline, according to the selected correlation method.

A fair agreement of predictions with several actual turbines experimental results was found in ref. [4], where also the applicability of different available correlations is discussed.

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LIST OF SYMBOLS

SYMBOLS

A	Area (in^2)
A_{CL}	Tip clearance area (in^2)
a	Throat opening of blade channel (in)
\bar{a}	Non-dimensional throat opening, a/a_m
C	Blade chord, see Fig. 6 (in)
c_l	Rotor tip clearance (in)
c_p	Specific heat at constant pressure ($\text{BTU}/\text{lb}_m\text{-}^\circ\text{R}$)
dA_n	Infinitesimal area, perpendicular to the flow (in^2)
E	Blade radius of curvature, see Fig. 6 (in)
\vec{f}_f	Frictional force per unit mass (lb/slug)
H	Total enthalpy (BTU/lb_m)
H	Boundary layer form factor
K	Boundary layer energy factor
K_{CL}	Coefficient, see eq. (31)
\vec{k}	Curvature coefficient, see eq. (16)
\vec{i}	Unit vector
i	Incidence angle
L	Mean distance between the stations 0 and 1 and 1 and 2 (in)
M	Mach number
m	Boundary layer exponent
\dot{m}	Mass flow rate (slug/sec)
\dot{m}_{CL}	Mass flow rate passing thru the rotor tip clearance (slug/sec)
NSETS	Number of conditions for which performance values have to be found
O	Blade opening (in)
P_T	Total pressure (psi)
p	Static pressure (psi)
q	Heat exchanged with external sources per unit mass (BTU/lb)
R_G	Gas constant ($\text{ft-lb}/\text{lb}_m\text{-}^\circ\text{R}$)
R_H	Radius at blade hub (in)
R_T	Radius at blade tip (in)
r	Radius (in)
S	Entropy ($\text{BTU}/\text{lb-}^\circ\text{R}$)
S^*	Non-dimensional entropy, S/c_p

T	Blade maximum thickness, see Fig. 6 (in)
T	Static temperature ($^{\circ}\text{R}$)
T_T	Total temperature ($^{\circ}\text{R}$)
TE	Thickness of blade trailing edge, in peripheral direction, see Fig. 6 (in)
TN	Thickness of blade trailing edge, in normal direction, see Fig. 6 (in)
t	Time (sec.)
u	Peripheral velocity (ft/sec)
V	Absolute velocity (ft/sec)
W	Relative velocity (ft/sec)
x	Non-dimensional radius, r/r_m
X_e	See eq. (30)
Y	Non-dimensional axial velocity, V_A/V_{Am}
Y	Total pressure loss coefficient, see eq. (40)
z	Number of blades

GREEK SYMBOLS

α	Absolute gas flow angle, see Fig. 5 (deg or radians)
β	Relative gas flow angle, see Fig. 5 (deg or radians)
β^*	Pressure ratio, see eq. (44)
β_o	Blade inlet angle, see Fig. 5 (deg or radians)
γ	Specific heat ratio
$\Delta\eta_{CL}$	Decrease in efficiency, due to rotor tip leakage, see eq. (47)
ΔR	Streamline displacement, see Fig. 1
δr	Streamline displacement, see Fig. 1
ζ	Kinetic energy loss coefficient
ζ^*	Kinetic energy loss coefficient used in the continuity equation, see eq. (28)
ζ_o	Kinetic energy loss coefficient for low Mach numbers, see eq. (42)
ζ_{CL}	Kinetic energy loss coefficient due to rotor tip leakage, see eq. (47)
η_{TS}	Total/static efficiency
η_{TT}	Total/total efficiency
θ	Angle between two radial planes (deg. or radians)
λ	Angle between the flow and the axis of the turbine in a meridional plane (deg. or radians)
μ	Gas viscosity (lb/sec-ft)
ξ	Area restriction factor, see eq. (27)
ρ	Gas density (lb _m /ft ³)
Φ	Non-dimensional flow function
X_p	Coefficient in Traupel's method of loss correlation, see eq. (46)
$\vec{\omega}$	Angular velocity (radians/sec)

SUBSCRIPTS

A	Axial
E	Equivalent
is	Isentropic expansion from total inlet conditions
m	Mean streamline
m	Meridional
R	Rotor
r	Radial direction, cylindrical coordinates
S	Stator
z	Axial direction, cylindrical coordinates
θ	Peripheral direction, cylindrical coordinates
o	Station ahead the stator
1	Station between stator and rotor
2	Station after the rotor

INTRODUCTION

An accurate knowledge of performance achievable with turbines of fixed geometry, both at design and off-design conditions, is important in advanced propulsion and power systems. The more accurate the prevision is, the more meaningful the optimization of the turbine itself and of the overall system will be.

A first way to solve this problem consists in the testing of a prototype, but it presents several disadvantages, namely it is expensive and time consuming. Furthermore, if effects of turbine geometry changes have to be investigated, different prototypes will be required.

An alternative solution to the problem consists in theoretical methods of prediction. The present availability of high speed computers allows one to solve systems of differential equations and to perform a large number of iterations in a short time. Moreover computer programs give the possibility of changing input data, so that a large variety of different turbine configurations and of working conditions can be investigated.

This report presents a computer program, derived by the three-dimensional calculation method given first by Vavra [1] and then developed by Eckert [2] and Harrison [3]. This method takes account of enthalpy and entropy gradients and of streamline curvatures and slopes in equations of motion, and of boundary layer thicknesses in continuity equation. It is applicable to single-stage axial turbines with unchoked flow.

A critical point in all theoretical performance calculations is the prediction of the losses that occur in the blade rows. In the present program, one can choose among five different loss correlation methods. For each method pertinent subroutines compute loss coefficients from blading geometry and flow characteristics. A comparative study of these methods can be found in [4], where they were applied to actual turbines of different design practices, in a wide range of all significant flow and geometrical properties.

The comparisons presented in [4] with experimental performance showed a fair agreement of predictions with all turbine stages that were examined.

The present report is divided into four sections: in the first the used equations are set up and necessary assumptions are discussed; in the second the method of solution is described; in the third different losses and discharge flow angles available prediction methods are described; and in the fourth the instructions for the use of computer program are given, with an example of input and output data. Eventually in the appendix the FORTRAN list of the MAIN routine and of the 28 subroutines is given.

METHOD OF ANALYSIS

The flow in turbomachines is ruled by the three fundamental laws of fluid dynamics, namely:

Equation of Motion

Energy Equation

Equation of Continuity.

All proper three-dimensional calculation methods proposed in the technical literature solve the differential equations derived by these laws. Differences among these methods are due to the three following reasons:

- different stations are chosen where equations are solved;
- different hypotheses are formulated to describe the real processes occurring in the machine;
- different numerical methods are applied in the solutions of the equations.

As far as the first problem is concerned in the present method, the equation of motion is applied at three stations, namely ahead of the stator, after the stator, and after the rotor, indicated respectively with subscripts 0, 1, 2 in Fig. 1. On the contrary, since blade geometry is given, the continuity equation is applied at blade throat sections 1* and 2* in Fig. 1, as suggested in [5].

those occurring in the boundary layers along the blade surfaces, and flow frictional losses between blades can therefore be neglected.

Assumption 4) is reasonable for uncooled blades, where energy changes due to heat transfer are negligible with respect to those related to velocity changes. If blades are cooled, the additional term q in the energy equation must be specified. Assumption 5) is valid in most cases. If the thermodynamic processes occur in ranges where real gas effects are not too strong, average values of gas properties give good approximations.

Some additional assumptions, regarding streamlines slope and curvature, are discussed in the section pertaining to the equation of motion.

As far as the third problem is concerned, the numerical small difference method was found to give a good convergence, and was therefore applied for the solution of the differential equations.

The following sections give the derivations of the equations with the above-listed assumptions.

Equation of motion

The general equation of motion for relative flow in vectorial form, is:

$$\frac{\partial \vec{W}}{\partial t} + \nabla H_R = \vec{W} \times (\nabla \times \vec{W} + 2\vec{\omega}) + T \nabla S + \vec{f}_f \quad (1)$$

$$\text{For absolute flow, where } \begin{cases} \vec{\omega} = 0 & (2) \\ \vec{V} = \vec{W} & (3) \\ H = H_R & (4) \end{cases}, \text{ equation (1)}$$

becomes:

$$\frac{\partial \vec{V}}{\partial t} + \nabla H = \vec{V} \times (\nabla \times \vec{V}) + T \nabla S + \vec{f}_f \quad (1a)$$

$$\text{With hypothesis } 1) \quad \frac{\partial \vec{W}}{\partial t} = 0, \quad \frac{\partial \vec{V}}{\partial t} = 0$$

$$\text{and hypothesis } 2) \quad \vec{f}_f = 0$$

This allows one to more accurately evaluate the pressure drops occurring in blade channels, since in this way it is possible to account for boundary layer thicknesses. This problem is particularly important at high Mach numbers where a small change in flow area causes large differences in pressure ratios.

As far as the second problem is concerned, the following assumptions are made:

- 1) Steady flow
- 2) Frictionless flow at the stations where the equations of motion are solved.
- 3) Axisymmetric flow at the stations where the equations of motion are solved.
- 4) Adiabatic flow.
- 5) The working fluid is a perfect gas, namely a gas having the equation of state:

$$p = \rho R_G T$$

and specific heat at constant pressure is independent of temperature and pressure.

Hypotheses 1) and 3) are interrelated since the flow can be steady in both rotor and stator only if it is axisymmetric. Although they are correct only for cascades with an infinite number of blades, axisymmetry and steady conditions are necessary assumptions since any other hypothesis would require unknown quantitative information on flow inside the blade rows. One problem connected with these assumptions is that downstream conditions cannot affect the flow upstream. Therefore pressure distributions at rotor exit can deviate from those imposed by the discharge conditions.

With assumption 2), all entropy changes are supposed to occur in the blade rows ahead of the considered station. This assumption can be justified by the fact that velocity gradients in the regions, where the equation of motion is applied, are in all probability smaller than

equations (1) and (1a) become respectively:

$$\nabla H_R = \vec{W} \times (\nabla \times \vec{W} + 2\vec{\omega}) + T \nabla S \quad (5)$$

$$\nabla H = \vec{V} \times (\nabla \times \vec{V}) + T \nabla S \quad (5a)$$

The following derivations for equation (5) are valid also for equation (5a), if substitutions (2), (3), (4) are performed.

It is convenient to express equation (5) in cylindrical coordinates (see Fig. 1 for symbols). Equating the three components of equation (5), the following equations are obtained:

a. tangential component:

$$\frac{1}{r} \frac{\partial H_R}{\partial \theta} = \frac{W_A}{r} \left[\frac{\partial W_A}{\partial \theta} - \frac{\partial(rW_u)}{\partial z} \right] - \frac{W_r}{r} \left[\frac{\partial(rW_u)}{r} - \frac{\partial W_r}{\partial \theta} \right] - 2\omega W_r + \frac{T}{r} \frac{\partial S}{\partial \theta} \quad (6)$$

b. meridional component:

$$\frac{\partial H_R}{\partial z} = W_r \left[\frac{\partial W_r}{\partial z} - \frac{\partial W_A}{\partial r} \right] - \frac{W_u}{r} \left[\frac{\partial W_A}{\partial \theta} - \frac{\partial(rW_u)}{\partial z} \right] + T \frac{\partial S}{\partial z} \quad (7)$$

c. radial component:

$$\frac{\partial H_r}{\partial r} = \frac{W_u}{r} \left[\frac{\partial(rW_u)}{\partial r} - \frac{\partial W_r}{\partial \theta} \right] - W_A \left[\frac{\partial W_r}{\partial z} - \frac{\partial W_A}{\partial r} \right] + 2\omega W_u + T \frac{\partial S}{\partial r} \quad (8)$$

with hypothesis 3) $\frac{\partial}{\partial \theta} () = 0$

and hypothesis 4) $\vec{w} \cdot \nabla H_R = 0$

the system of equations (6), (7), (8) reduces to equation (9):

$$\frac{\partial H_R}{\partial r} = \frac{W_u}{r} \frac{\partial(rW_u)}{\partial r} - W_A \frac{\partial W_r}{\partial z} + W_A \frac{\partial W_A}{\partial r} + 2\omega W_u + T \frac{\partial S}{\partial r} \quad (9)$$

$$\text{Using substitution: } H_E = H_R + \frac{u^2}{2} \quad (10)$$

with hypotheses 5) $H = c_p T$ and 4) $H_E = \text{const}$ (see Fig. 2)

$$T_2 = \frac{H_E}{c_p} - \frac{w_2^2}{2c_p} \quad (11)$$

and introducing the non-dimensional quantities:

$$Y = \frac{W_A}{W_{Am}} \quad (12)$$

$$X = \frac{r}{r_m} \quad (13)$$

$$S^* = \frac{S}{c_p} \quad (14)$$

Equation (9) becomes:

$$\begin{aligned} & \frac{1}{Y^2} \frac{\partial Y^2}{\partial x} + \cos^2 \beta \left(- \frac{2r_m}{W_A} \frac{\partial W_r}{\partial z} - \frac{1}{\cos^2 \lambda} \frac{\partial S^*}{\partial x} \right) + 2 \tan \beta \frac{\partial \beta}{\partial x} \\ & + \frac{2}{x} \sin^2 \beta + \frac{4 \sin \beta \cos \beta}{W_{Am} Y} + \frac{2 U_m U \cos^2 \beta}{W_A^2 Y^2} - \frac{2 \cos^2 \beta}{W_{Am}^2 Y^2} \frac{\partial H_E}{\partial x} + \\ & + \left[\frac{2 H_E \cos^2 \beta}{W_{Am}^2 Y^2} - \sin^2 \beta \right] \frac{\partial S^*}{\partial x} = 0 \end{aligned} \quad (15)$$

In this equation, terms containing $\cos \lambda$ and $\frac{\partial W_r}{\partial z}$ depend respectively on streamlines slope and curvature. Vavra [1] showed that the most reasonable assumptions, when only three points of a streamline are known, are the following (see Fig. 1 for symbols):

$$\frac{1}{W_A} \frac{\partial W_r}{\partial z} = + k \frac{\delta_r}{L^2} \quad (16)$$

$$\cos^2 \lambda = \frac{L^2}{L^2 + \left(\frac{\Delta R}{2}\right)^2} \quad (17)$$

In equation (16) the positive sign holds for station (2) and the negative one for station (1), for δr positive as shown in Fig. 1. Suggested values for k are between 4 and 6. Finally with equations (16) and (17) into (15) the following equation is obtained:

$$\begin{aligned} \frac{d(\ln Y^2)}{dx} = & -\cos^2\beta \left[2k r_m \frac{\delta r}{L^2} - \frac{L^2 + (\frac{\Delta R}{2})^2}{L^2} \frac{dS^*}{dx} \right] \\ & - 2 \tan \beta \frac{d\beta}{dx} - \frac{2}{x} \sin^2\beta - \frac{4 U_m \sin\beta \cos\beta}{W_{Am} Y} - \\ & - \frac{2 U_m U \cos^2\beta}{W_{Am}^2 Y^2} + \frac{2 \cos^2\beta}{W_{Am}^2 Y^2} \frac{dH_E}{dx} - \left[\frac{2 H_E \cos^2\beta}{W_{Am}^2 Y^2} - \sin^2\beta \frac{dS^*}{dx} \right] \end{aligned} \quad (18)$$

For the station after the stator row, the following equation can be derived with (2), (3) and (4) in equation (18):

$$\begin{aligned} \frac{d(\ln Y^2)}{dx} = & -\cos^2\alpha \left[(-2K r_m \frac{\delta r}{L^2}) - \frac{L^2 + (\Delta R/2)^2}{L^2} \frac{dS^*}{dx} \right] - \\ & - 2 \tan \alpha_1 \frac{d\alpha}{dx} - \frac{2}{x} \sin^2\alpha_1 + \frac{2 \cos^2\alpha}{Y^2 V_{Am}^2} \frac{dH}{dx} - \left[\frac{2 H \cos^2\alpha}{Y^2 V_{Am}^2} - \right. \\ & \left. - \sin^2\alpha_1 \right] \frac{dS^*}{dx} \end{aligned} \quad (18a)$$

Equation(18a) could also have been derived directly from equation (5a).

In equations (18) and (18a) the last term is depending on flow irreversibilities due to frictional losses. The specific entropy S^* will be calculated, after [1], by:

$$S^* = \ln \left[\frac{\frac{Y^2 W_{Am}^2}{2 H_E \cos^2\beta}}{Y W_{Am}^2} \right] + S_E^* \quad (19)$$

$$1 - \frac{2 H_E \cos^2\beta (1-\zeta)}{Y W_{Am}^2}$$

and by:

$$S^* = \ln \left[\frac{\frac{Y V_{Am}^2}{2 H_O \cos^2 \beta}}{\frac{Y V_{Am}^2}{2 H_O \cos^2 \beta (1 - \zeta)}} \right] + S_O^* \quad (19a)$$

where ζ represents the kinetic energy loss coefficient which is established by the chosen loss correlation method.

Energy equation

The energy equation in a relative flow is given by:

$$\vec{W} dt \cdot \nabla (H_R) = \vec{W} dt \cdot \nabla (q) - \frac{\partial}{\partial t} \left(\frac{W^2}{2} \right) dt \quad (20)$$

and in an absolute flow, with (2), (3) and (4) in (20), :

$$\vec{V} dt \cdot \nabla (H_R) = \vec{V} dt \cdot \nabla (q) - \frac{\partial}{\partial t} \left(\frac{V^2}{2} \right) dt \quad (20a)$$

where q is the heat added to a unit mass particle from external sources. With assumptions 1) $\partial/\partial t () = 0$ and 4) $q = 0$, equations (20) and (20a) become:

$$\vec{W} dt \cdot \nabla (H_R) = 0 \quad (21)$$

$$\vec{V} dr \cdot \nabla (H) = 0 \quad (21a)$$

Equations (21) and (21a) indicate that for steady, adiabatic flow the relative total enthalpy is constant along any streamline. This does not mean that $\nabla (H_R) = 0$, since relative total enthalpy can assume different values for different streamlines.

Equation of continuity

Pressure ratios across a stage necessary to discharge a given flow rate are function of inlet Mach number, blade openings and the losses occurring in the blades. For an isentropic flow the mass flow rate passing thru a section dA_n perpendicular to the flow is given by:

$$d \dot{m}_{is} = \rho_{is} V_{is} dA_n \quad (22)$$

where ρ_{is} and V_{is} are respectively the density and the velocity corresponding to an isentropic process. It can be shown that equation (22) can be expressed for a perfect gas (see assumption 5) by:

$$d \dot{m}_{is} = \frac{P_T}{\sqrt{R_G T_T}} \Phi dA_n \quad (23)$$

where P_T , T_T are, respectively, total pressure and total temperature (constant for isoentropic and adiabatic flows) and Φ is a function of pressure ratio and gas isentropic exponent, namely:

$$\Phi = \sqrt{\frac{2\gamma}{\gamma-1} \left[\left(\frac{P}{P_T}\right)^{2/\gamma} - \left(\frac{P}{P_T}\right)^{\frac{\gamma+1}{\gamma}} \right]} \quad (24)$$

With the symbols in Fig. 3, assuming the pressure to be constant along the throat "a" equation (23) becomes:

$$d \dot{m}_{is} = \frac{P_T}{\sqrt{R_G T_T}} \Phi a \cos \lambda dr \quad (25)$$

Integrating equation (25) along the radius and for z blades the flow rate passing thru the stage is:

$$\dot{m}_{is} = z \int_{R_H}^{R_T} \frac{P_T}{\sqrt{R_G T_T}} \Phi a \cos \lambda dr \quad (26)$$

In actuality, the flow rate will be less than \dot{m}_{is} , because of the losses occurring in the blade channels.

A so-called restriction factor:

$$\xi = \frac{d \dot{m}}{d \dot{m}_{is}} \quad (27)$$

is now introduced.

In [5] the following relationship between ξ and the kinetic energy loss coefficient ζ^* is derived:

$$\xi = \frac{1}{1 + \zeta^* \frac{H}{K}} \quad (28)$$

where H and K are respectively the boundary layer wake form factor and the energy factor. For boundary layer profiles following the power law, the ratio $\frac{H}{K}$ can be expressed in power series:

$$\frac{H}{K} = \frac{1 - (1-Xe) \sum_{i=1}^{\infty} \left[Xe^{i-1} / 1+(2i-1)m \right]}{(1-Xe) \sum_{i=1}^{\infty} \left[Xe^{i-1} / 1+(2i-1)m \right] - (1-Xe) \sum_{i=1}^{\infty} \left[Xe^{i-1} / 1+(2i+1)m \right]} \quad (29)$$

where:

$$Xe = 1 - (P/P_T)^{\gamma-1/\gamma} \quad (30)$$

and m is the exponent of the power law for the boundary layer profile (for incompressible flow, m depends on Reynolds number and varies between 1/7 and 1/10). Experience has shown that the loss coefficient ζ^* has to be taken less than the value ζ_{TOT} that corresponds to the overall energy losses that are given by the correlation methods. In the present program one of the three following different assumptions for ζ^* can be chosen: 1) $\zeta^* = \zeta_{TOT}$; 2) $\zeta^* = \frac{1}{2}\zeta_{TOT}$; 3) $\zeta^* = \zeta_{PROFILE}$.

While assumption 1) is certainly conservative, it is difficult to state which of assumptions 2) or 3) is the best one. Good agreement between experimental values and predicted flow rates for all considered turbines at low tip clearance values was found in [4], where assumption 2) was adopted. Assumption 3) was however found in closer agreement with test data for large values of rotor tip clearances.

Tip leakage flow rate will be calculated by using the relation:

$$\dot{m}_{CL} = \frac{1}{K_{CL}} \left(\xi \frac{P_T}{\sqrt{R T_T}} \Phi \cos \lambda \right)_{TIP} A_{CL} \quad (31)$$

$$\text{where: } A_{CL} = 2\pi R_H c \ell \quad (32)$$

Since the flow is not perpendicular to the leakage area, and since the restriction factor ξ_{TIP} does not account for boundary layer blockage occurring on shroud, the leakage flow rate must be divided by a coefficient K_{CL} (larger than 1), assumed equal to 2 in the present program.

With equations (29) and (33), the overall flow rate passing thru a blade row is given by:

$$\dot{m}_{TOT} = \dot{m} + \dot{m}_{CL} = z \int_{R_H}^{R_T} \xi \frac{P_T}{\sqrt{R_G T_T}} \Phi a \cos \lambda dr + \frac{1}{K_{CL}} \left(\xi \frac{P_T}{\sqrt{R T_T}} \Phi \cos \lambda \right)_{TIP} A_{CL} \quad (33)$$

Equation (33) can be expressed in the following dimensionless form:

$$\dot{m} = z R_m a_m \int_{x_H}^{x_T} \left(\xi \frac{P_T}{\sqrt{R_G T_T}} \Phi \bar{a} \cos \lambda \right)_x dx + \frac{2\pi}{K_{CL}} \xi_{TIP} \frac{x_T}{x_m} \frac{c \ell}{z \bar{a}} \left(\frac{P_T}{\sqrt{R T_T}} \Phi \cos \lambda \right)_{TIP} \quad (34)$$

$$\text{where } \bar{a} = \frac{a}{a_m} \quad (35)$$

METHOD OF SOLUTION

Since no direct solution of governing equations is possible, an iterative method is followed. At first, solutions are found without considering the influence of streamline curvature and slope. The derived radial shifts are then used for the second and final cycles of calculation. In Fig. 4 the scheme followed in the computer program is indicated. The names in brackets correspond to the names of the subroutines which perform the operations described next to the brackets.

The flow ahead of the stator is supposed to be uniform, so that the radii of streamlines dividing the flow rate in equal parts can be easily computed. For stator and rotor rows the following functions are supposed to be known:

STATOR	ROTOR
$\alpha_1 = \alpha_1 (r, \text{flow condition})$	$\beta_2 = \beta_2 (r, \text{flow condition})$
$a_1 = a_1 (r)$	$a_2 = a_2 (r)$
$\zeta_s = \zeta_s (r, \text{flow condition})$	$\zeta_R = \zeta_R (r, \text{flow condition})$

All derivatives used in the following analysis are computed by finite difference methods. The computer program calculates flow conditions along 5 streamlines. However the same scheme of calculation can be applied to any number of streamlines. By assuming the inlet Mach number M_0 , the flow rate passing thru section 0 can easily be determined. Radial position of streamlines at station 1, R_1 , and the axial velocity at mean streamline, V_{Alm} , are also assumed a priori, and will be checked later.

For any given values of V_{Alm} , ΔR , and δr , equation (18a) can be solved by iteration. In fact it can be reduced to:

$$\frac{d(\ln Y^2)}{dx} = I(x) \quad (36)$$

Equation (36) integrated, with boundary condition $\begin{cases} x = 1 \\ y = 1 \end{cases}$ gives: (37)

$$\ln Y^2 = \int_1^x I dx \quad (38)$$

and

$$Y = e^{\frac{1}{2} \int_1^x I dx} \quad (39)$$

Solutions are found by trial and error, assuming approximate values of $Y(x)$ and verifying equation (39). Once equation (39) has been solved, all flow conditions at station (1) are known and the overall continuity equation (34) can be solved. If the calculated flow rate is different from the assumed one, V_{aml} will be changed and the iteration shown in Fig. 4 is performed. When overall continuity equation is satisfied, the assumed radial position of streamlines is checked, and necessary iterations are performed. If streamlines continuity is met also, the flow at the rotor inlet is known completely. A similar iterative method is then followed for station 2 after the rotor as indicated in Fig. 4. If the flow after the rotor is known also, the radial shift of the streamlines can be determined, and the whole calculating procedure is repeated by taking account of the streamlines curvatures and the slopes in solving the equation of motion.

Eventually the resulting average pressure ratio across the turbine is compared with the desired one, and if the difference is larger than a small prefixed error, the complete calculating scheme is repeated again with a new value of inlet Mach number. After the specified pressure ratio has been obtained, all output information is calculated and printed.

FLOW ANGLES AND LOSS COEFFICIENTS CORRELATIONS

Evidently, even the most accurate method of performance prediction will give results in disagreement with actuality, if the flow angles and/or the losses in the blade rows are not predicted with the necessary accuracy.

Several correlations, either empirical or theoretical, have been proposed in the technical literature. A comparative study of five correlations methods available in the present program can be found in Reference [4]. The purpose of the present chapter is not to discuss the applicability of these different correlations, but only to indicate the problems which arose, in applying the different correlation methods. In Fig. 5 the nomenclature and sign convection of the velocity triangles is indicated. Unless otherwise stated, the variables are positive if directed as in Fig. 5. In Fig. 6 the blading geometry characteristics used in the different correlation methods are indicated. The names

given in Fig. 6 are the FORTRAN names used in the program for stator quantities at the mean radius.

The program is organized in such a way that other than considered flow angles and loss coefficients correlations can be introduced by adding a few cards and pertinent subroutines. Since the solution method is iterative, added subroutines can use blading and flow characteristics that are not used in the present program.

a) Flow angles correlations

Two methods, one given by Ainley and Mathieson [6] and the other given by Traupel [10] are available in subroutines. In both these methods, exit flow angles (α_1, β_2) are considered to be independent on inlet angles (α_0, β_1). While in Traupel's method the angles are a function of blading geometry only, in A&M method also the exit Mach number is supposed to affect the angles. Although both these methods are given for a "reference diameter approach", they will be applied in three-dimensional calculations with the assumption that at any radius the angle is related only to the corresponding characteristics. Since during the calculations both Mach numbers and streamlines radii will be changed, flow angles corrections are automatically performed during the solution of the equation of motion.

b) Loss coefficients correlations

The assumption is made that at any particular radius the loss coefficient is a function of the characteristics at that radius only. It is realized that this assumption is not realistic, since secondary and leakage losses are certainly not equally distributed along the radius. On the other hand, secondary and tip leakage flows are not understood well enough at the present time to formulate any other reasonable assumption.

Five loss correlation methods given in Ref. [6-10] are available in subroutines. They calculate loss coefficients from given blade geometry and flow angles and velocities.

The first considered method is the Ainley and Mathieson [6] method. Recently an improvement to the original method was given by Dunham and Came [7]. Both methods are available in subroutines. These methods express losses by total pressure loss coefficient γ ,

defined by:

$$Y = \frac{P_{T0} - P_t}{P_t - P} \quad (40)$$

and assume Y to be independent on pressure ratio P_{T0}/P . The relationship between the kinetic energy loss coefficient ζ and Y is given by:

$$\zeta = \frac{\left[\frac{1+Y}{1+Y \cdot P/P_{T0}} \right]^{\frac{\gamma-1}{\gamma}} - 1}{\left(\frac{1}{P/P_{T0}} \right)^{\frac{\gamma-1}{\gamma}} - 1} \quad (41)$$

The assumption that Y is constant implies that ζ decreases with increasing Mach number. Since this behavior is not experimentally proved, two alternatives to equation (41) are available in the program. In both these alternatives ζ is supposed to be constant with pressure ratio, and assumes respectively the value corresponding to very low Mach numbers (eq. 42) or to an exit Mach number of 0.8 (eq. 43).

$$\zeta = \zeta_0 = \lim_{P/P_{T0} \rightarrow 1} \zeta = \frac{Y}{1+Y} \quad (42)$$

$$\zeta = \zeta_{M=0.8} = \frac{\left[\frac{1+Y}{1+Y \cdot \beta^*} \right]^{\frac{\gamma-1}{\gamma}} - 1}{\left(\frac{1}{\beta^*} \right)^{\frac{\gamma-1}{\gamma}} - 1} \quad (43)$$

where β^* is the pressure ratio corresponding to an isentropic Mach number of 0.8, given by:

$$\beta^* = \left[1 + \frac{\gamma-1}{2} \times .8^2 \right]^{-\frac{\gamma}{\gamma-1}} \quad (44)$$

The same alternatives are given also for the simple method of Lenherr and Carter [9].

For the method of Balje' [8], the boundary layer wake form factor H must be introduced.

With the same nomenclature and assumptions as in eq. (29), H can be expressed by:

$$H = \frac{1 - (1-X_e) \sum_{i=1}^{\infty} \left(\frac{X_e^{i-1}}{1+m(2i-1)} \right)}{(1-X_e) \sum_{i=1}^{\infty} \left(\frac{X_e^{i-1}}{1+m(2i)} \right) + (1-X_e) \sum_{i=1}^{\infty} \left(\frac{X_e^{i-1}}{1+m(2i-1)} \right)} \quad (45)$$

In this program the assumption $m = 0.15$ is made. If another exponent has to be introduced, card #17 must be changed in the MAIN routine.

In Traupel's method, the loss coefficients depend on the coefficient:

$$X_p = X_p (Re, \text{relative surface roughness, turbulence factor}). \quad (46)$$

In the present program, the standard value of $X_p=1$ is assumed. If the considered turbine has peculiar values of surface roughness or Reynolds numbers, it will be necessary to change card #15 in the MAIN routine.

For conformity with the other methods, tip clearance losses are expressed by the kinetic energy loss coefficient ζ_{CL} , instead of by the overall efficiency decrease $\Delta\eta_{CL}$ as in [10]. The relationship between $\Delta\eta_{CL}$ and ζ_{CL} is given by:

$$\zeta_{CL} = (1-\zeta_r) \times \left[1 - \left(1 - \frac{\Delta h_{is} \Delta\eta_{CL}}{u_2 W_{w2}} \right)^2 \right] \quad (47)$$

where Δh_{is} is the isentropic enthalpy drop through the turbine and ζ_r is the rotor kinetic energy loss coefficient for zero tip clearance.

COMPUTER PROGRAM

The present computer program is composed of a MAIN routine and 28 subroutines, eight of which are concerned with flow calculations, two with flow angle correlations, 15 with blade loss correlations, and three with numerical interpolations. A brief description of the purpose of each routine is given below.

a) MAIN routine

1. The primary function of the MAIN routine is to control the overall logic of the computer program which is indicated in Fig. 4. Furthermore the input data are read, numerical constants are introduced, interpolations concerning stall and blade angles and flow areas are performed and the flow angles are computed. Eventually, also the output data are printed.

b) Flow calculation subroutines

2. In subroutine CHAN the flow rate passing through the section 0 is calculated from the conditions ahead of the stator.

3. In subroutine STAT~~OR~~ the equation of motion (18a) is solved, and the flow conditions after the stator are calculated. If necessary, new values of the flow angles are also calculated.

4. In subroutine ROT~~OR~~1 the absolute flow properties calculated in STAT~~OR~~ are converted into relative properties at rotor inlet.

5. In subroutine ROT~~OR~~2 the equation of motion (18) is solved, and the flow conditions after the stator are calculated. If necessary, also new values of flow angles are calculated.

6. In subroutine FL~~OW~~R the flow rate passing through the blade sections, either at station 1 or 2, is computed by equation (34), and compared with the value calculated in CHAN. If the difference is larger than a small prefixed error, a new value of the axial velocity at the mean streamline is computed. The fractional values of the flow rate passing between the hub and each streamline are calculated also.

7. In subroutine SLINE the fractional values of the flow rate computed in FL~~OW~~R are compared with prefixed ones. If the differences are larger than a small specified error, new values of the streamline radii, either at station 1 or 2, are computed.

8. In subroutine ALØS1 the stator loss coefficients are calculated from the chosen loss correlation, as a function of blading geometry and flow characteristics. Coefficients ζ^* of (28) are computed for the stator blades.

9. In subroutine ALØS2 the rotor loss coefficients are calculated from the chosen loss correlation as function of blading geometry and flow characteristics. Coefficients ζ^* of equation (28) are computed for the rotor blades.

c) Flow Angles Subroutines

10. In subroutine ANGAIN the flow angles are computed from blading geometry and exit Mach numbers with the method of Ainley and Mathieson [6].

11. In subroutine ANGTRA the flow angles are computed from blading geometry with Traupel's method [10].

d) Loss Correlation Subroutines

The six subroutines given below are concerned with the methods [6] and [7].

11. In subroutine CØEFFI the input cards concerning the profile losses of nozzle blades given in [6] are read, and the necessary interpolations are performed.

12. In subroutine CØEFF2 a similar process is followed for the profile losses of impulse blades.

13. In subroutine CALYAØ the profile losses of nozzle blades are calculated with the coefficients computed in CØEFFI.

14. In subroutine CALYA2 the profile losses of impulse blades are calculated with the coefficients computed in CØEFF2.

15. In subroutine FIG6 the necessary interpolations for calculating the secondary losses are performed.

16. In subroutine AINLØS the overall loss coefficients with either method [6] or [7] are calculated from blading geometry and flow angles. Subroutine AINLØS must be called after subroutines CØEFFI and CØEFF2, and calls subroutines CALYAØ, CALYA2 and FIG6.

The eight subroutines given below are concerned with method (11) of the loss correlation.

17. In subroutine TRAUP1 the input cards concerning the profile losses given by Traupel's method are read, and the necessary interpolations are performed.

18. In subroutine XPØ the basic profile loss coefficients are computed with the coefficients calculated in TRAUP1.

19. In subroutine CSIM Mach number effects on the loss coefficients are computed.

20. In subroutine CID the effects due to the trailing edge thickness and nontwisted blades on the loss coefficients are computed.

21. In subroutine CSIW the end wall loss coefficients are computed.

22. In subroutine CSIR the secondary loss coefficients are computed.

23. In subroutine ALEAK the tip leakage efficiency losses are computed.

24. In subroutine TRAUP2 the overall loss coefficients are computed with Traupel's method [10] from both blade geometry and flow characteristics. Subroutine TRAUP2 has to be called after TRAUP1, and calls subroutines XPØ, CSIM, CID, CSIW, CSIR and ALEAK.

25. In subroutine ZALIØS the overall loss coefficients are computed with the method of Lenherr and Carter [9] from the flow angles and the relative velocities.

26. In subroutine BALJE the overall loss coefficients are computed with Balje's method [8] from blade geometry and flow conditions.

e) Numerical Interpolation Subroutines

27. In subroutine PARAB the coefficients of the parabolic interpolations used to approximate curves are calculated.

28. In subroutine CHBFT the coefficients of the polynomial of arbitrary degree, which is the best fit of a given array of points in the Chebyshev sense, are calculated.

29. In subroutine YC the value of the polynomial determined in CHBFT, corresponding to a given value of the independent variable, is calculated.

Description of Input Data

The input data of the present computer program consist of 63 + NSETS cards, NSETS being the number of conditions for which performance values are to be obtained. They can be divided in three sections. In the first, the constant values used in the methods of Ainley and Mathieson and of Traupel are introduced (these 36 cards are read respectively in subroutines CØEFFI, CØEFF2 and TRAUP1). In the second, the specifications about the detail of output desired, the tolerances of the numerical solutions, the systems of correlation of the flow angles and the loss coefficients, the coefficients used in the equation of continuity, the gas properties and the turbine geometry are stated. In the third, the conditions for which performance values are to be found are introduced.

The 36 input cards for the first section are given in Table I. They will remain unchanged for all turbines. Detailed description of the 27 cards composing the second section of the input are given in Table II, where also a description of the third input section can be found in cards #28 and #29. An example of the input cards described in Table II is given in Table III.

Description of the normal output

In Table IV the output corresponding to the input of Tables I and III is given. It can be seen that it consists of 1 + 3 NSETS sheets. In the first sheet the turbine geometrical dimensions and the choice among the different correlation methods are described. Then three sheets for every set of operating conditions are printed. The first of these sheets describes the flow conditions at station 1 after the stator row; the second one the flow conditions after the rotor row; and the third one gives the main overall turbine characteristics and the mass averaged performance values.

ACKNOWLEDGEMENTS

The author would like to express his appreciation to Dr. M. H. Vavra for providing guidance and counsel during the investigation.

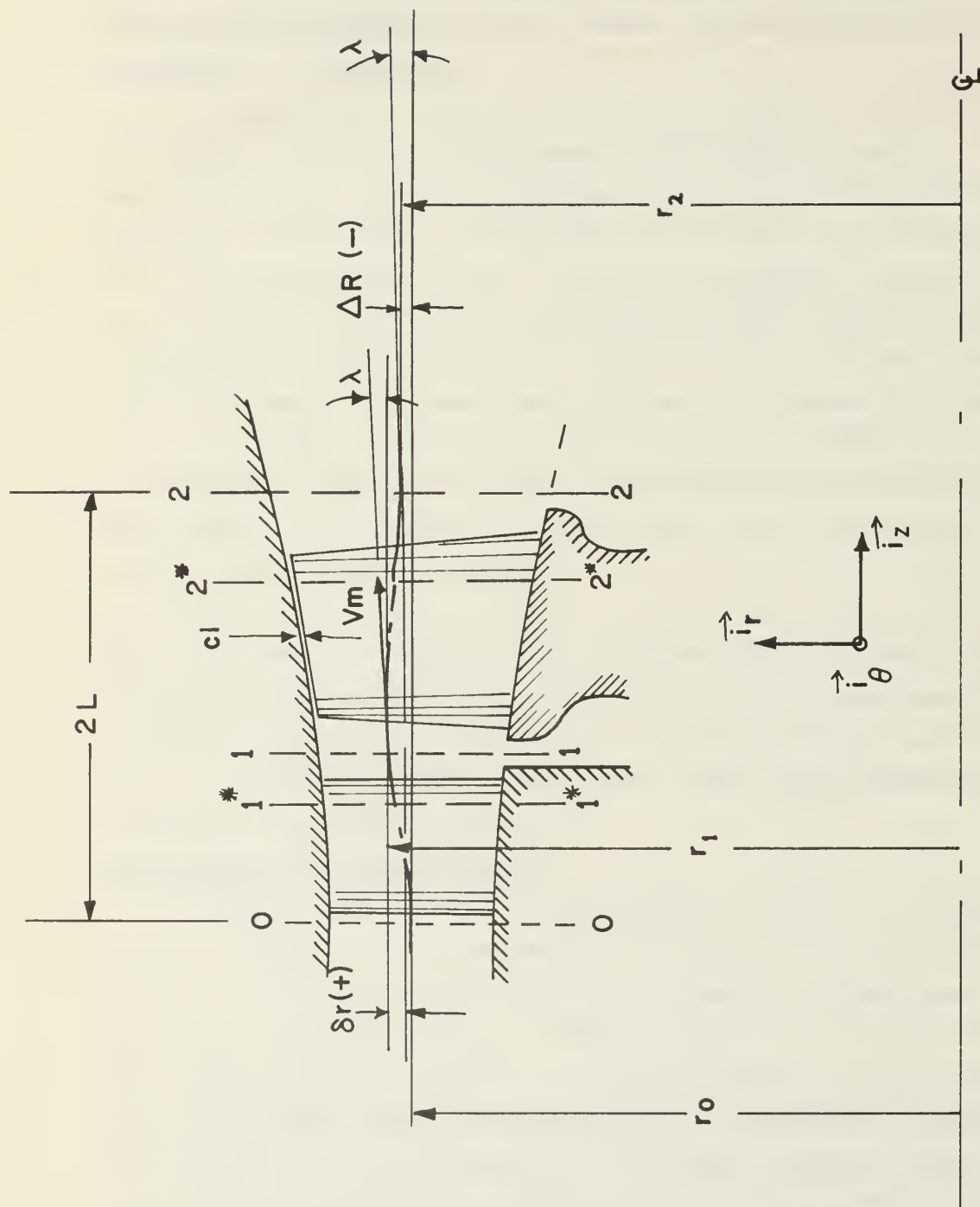


Fig . 1 Coordinates , stations and streamline nomenclature.

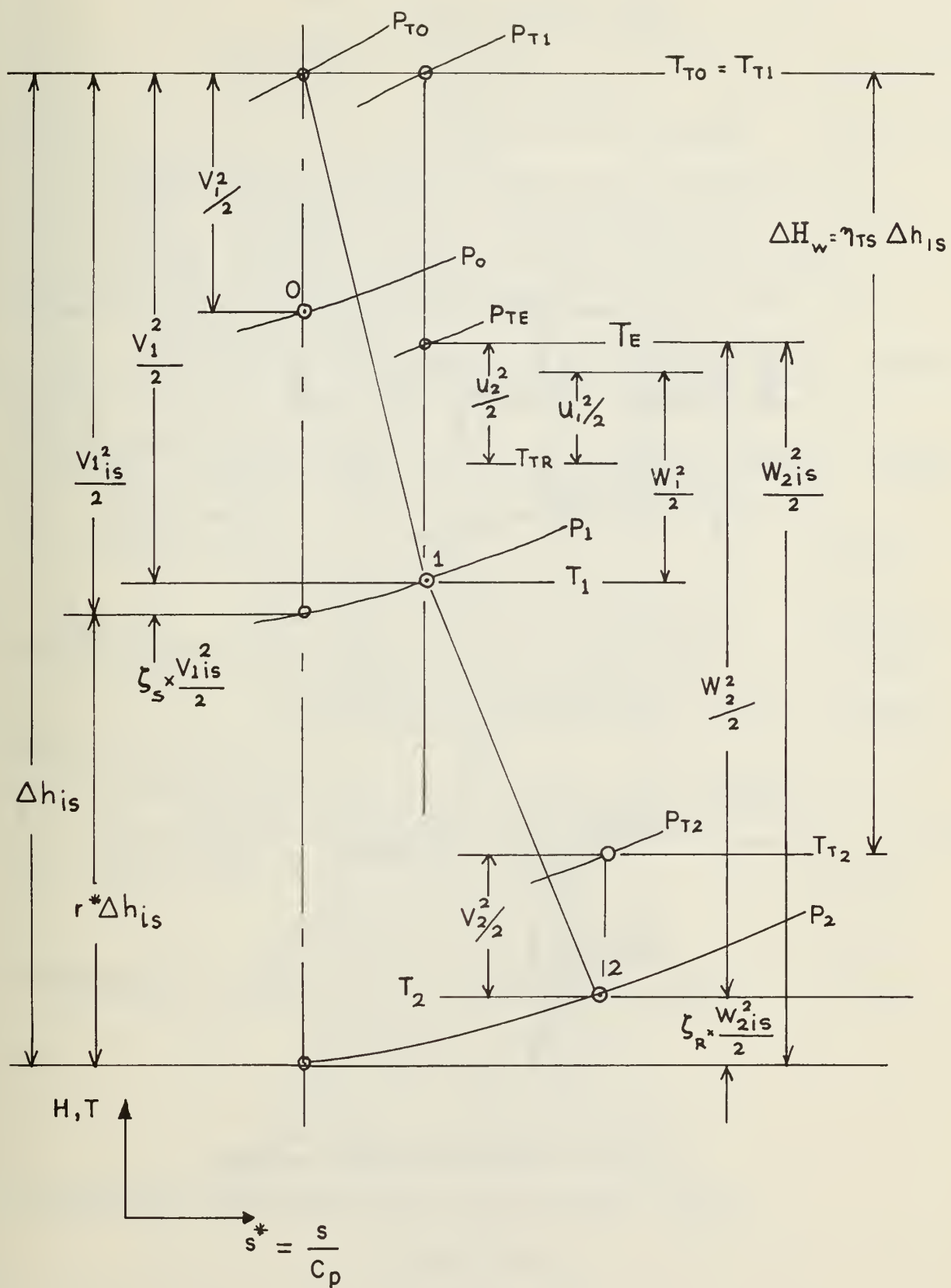


Fig. 2— Turbine thermodynamic process.



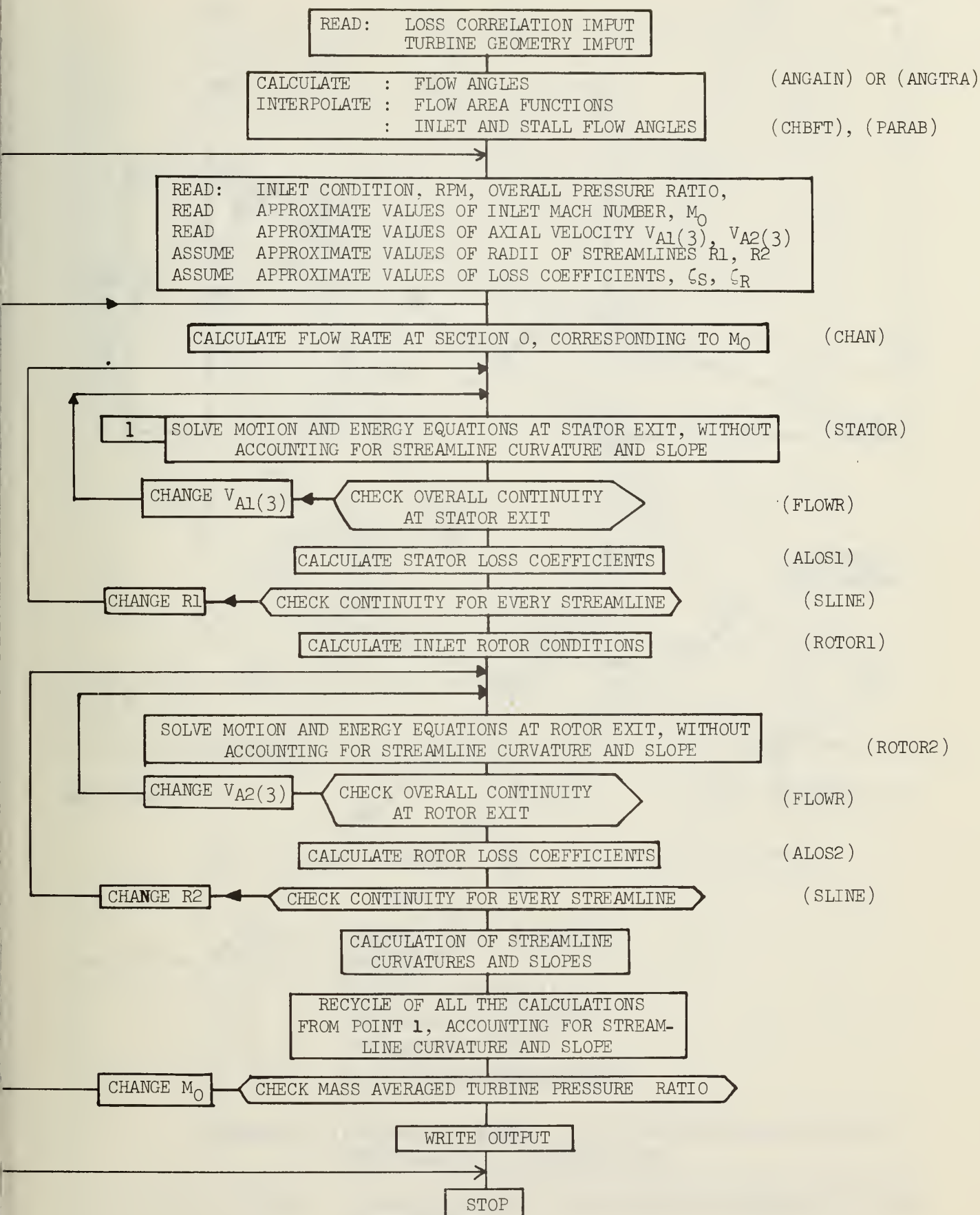


Fig.4 Flow diagram

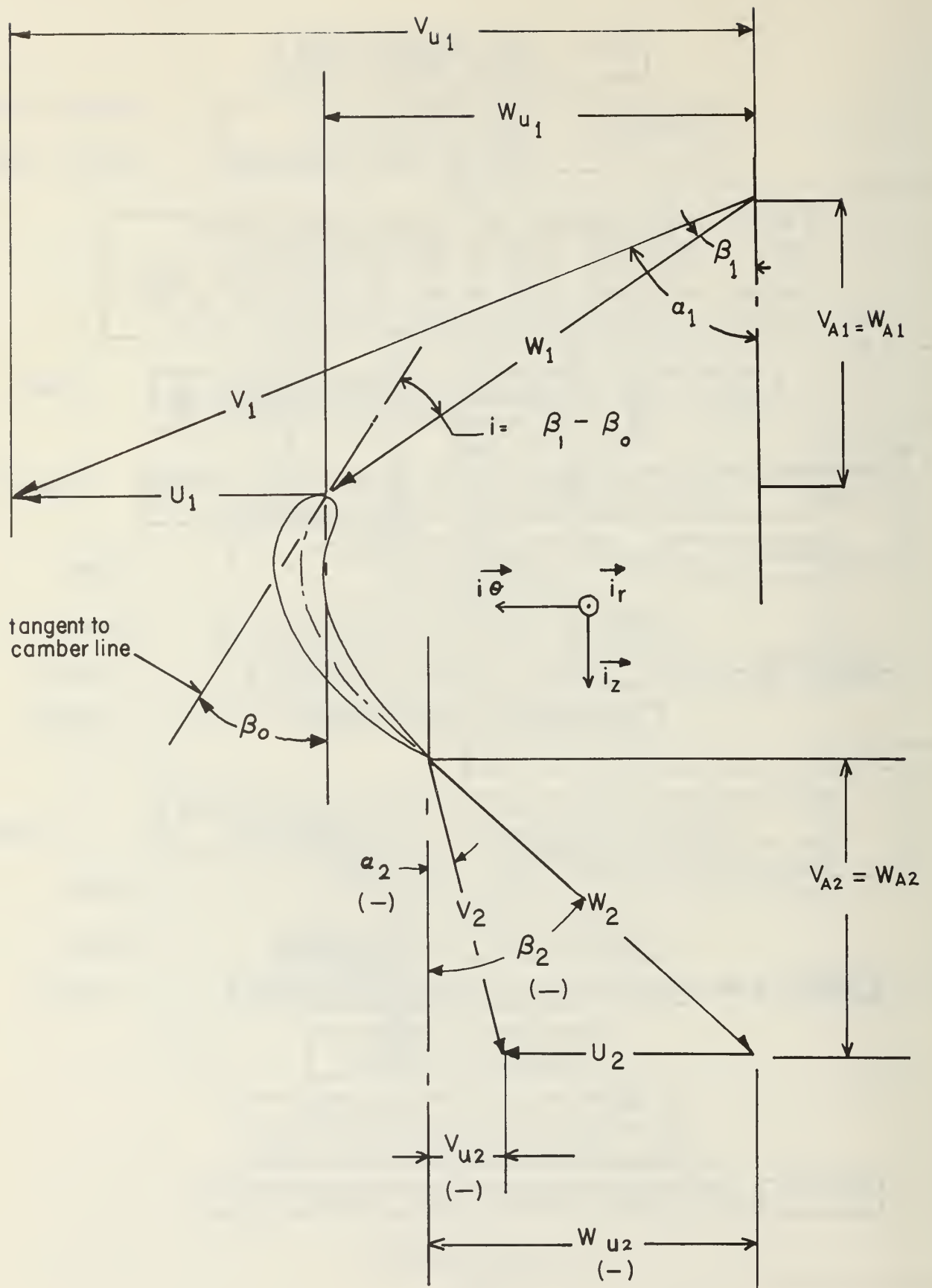
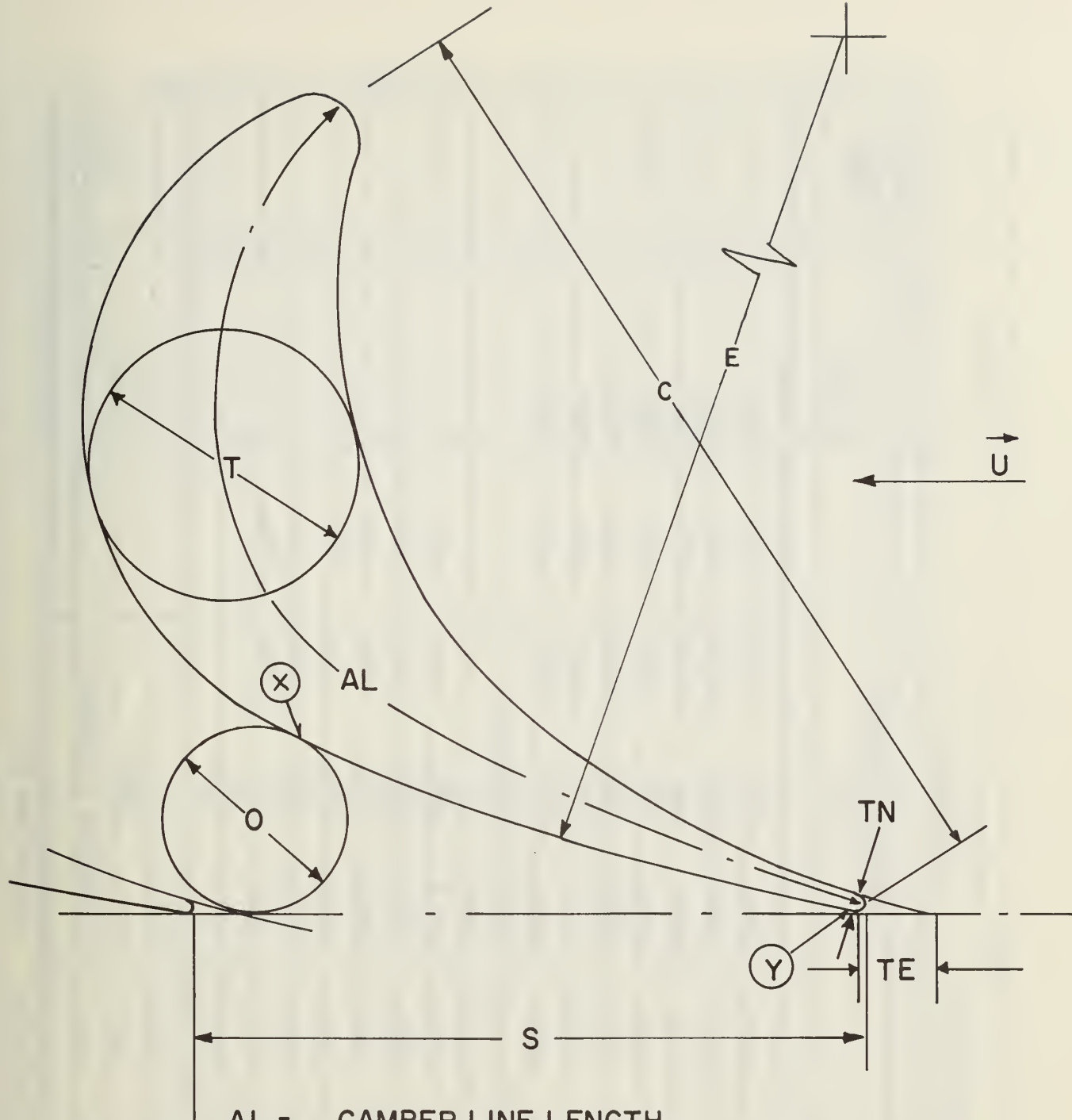


Fig.5 Velocity diagrams



AL = CAMBER LINE LENGTH

C = CHORD

O = THROAT DIAMETER

E = CURVATURE RADIUS

S = SPACING

T = MAXIMUM THICKNESS

TE = TRAILING EDGE THICKNESS PROJECTED IN PERIPHERAL DIRECTION

TN = TRAILING EDGE THICKNESS, NORMAL TO FLOW DIRECTION

Fig.6 Blade geometry nomenclature

PROGRAMMER E. MACCHI	PROGRAM AXIAL TURBINES	DATE AUG. 1970
SPECIAL INSTRUCTIONS		
PAGE 1	OF 2	
CARD PUNCH OPERATOR		
DATE COMPLETED		
FIXED INPUT DATA		
SUBMIT FOR PROCESSING		YES <input type="checkbox"/> NO <input type="checkbox"/>

STATE- MENT NO.		FORTAN STATEMENT																														SERIAL NUMBER																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																		
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1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																			

TABLE I (1)

[illegible]

EIGHTY COLUMN FORTRAN CODING SHEET 12ND PGS 5220/1 (11-66) TABLE I (2)

CARD	FORTRAN NAME	FORMAT	DESCRIPTION	COMMENTS
1	IND INZ IWR	I 5 I 5 I 5	PRINT INDEX PRINT INDEX PRINT INDEX	SEE TABLE IV
2	ICØR IAI IAN ICL IINC ICØZ ICØN	I 5 I 5 I 5 I 5 I 5 I 5 I 5	LOSS INDEX LOSS INDEX ANGLE INDEX CLEARANCE INDEX LOSS INDEX LOSS INDEX CONTINUITY INDEX	SEE TABLE V
3	VIS2 VIS3 CP EMME GAM	E 12.3 E 12.3 F 10.4 F 10.4 F 10.4	VISCOSITY AT STATION 1 VISCOSITY AT STATION 2 SPECIFIC HEAT AT P=CONST MOLECULAR MASS ISOENTROPIC EXPONENT	(LB/SEC-FT) (LB/SEC-FT) (BTU/LB-DEG. F)
4	TIPC ZS ZR	F 8.4 F 8.4 F 8.4	ROTOR TIP CLEARANCE STATOR NUMBER OF BLADES ROTOR NUMBER OF BLADES	(INCHES)
5	CV CK	F 8.4 F 8.4	AXIAL DISTANCE BETWEEN STATIONS CURVATURE FACTOR	SEE EQ. (16) AND (17) SEE EQ. (16)
7	STALL (I), I=1,10	10F 5.2	ROTOR STALL ANGLES AT 10 RADII EQUALLY SPACED FROM HUB TO TIP	DEFINED BY AINLEY METHOD [6]
6	BETAØ(I) I=1,5,2	3F 8.4	ROTOR BLADE ANGLES, AT HUB, MEAN AND TIP SECTIONS	DEFINED BY THE TANGENT CAMBER LINE SEE FIG. 6

TABLE II (1)

CARD	FORTRAN NAME	FORMAT	DESCRIPTION	COMMENTS
8	AL (I), I=1,5	5F8.4	STATOR BLADE OPENINGS AT FIRST FIVE RADII OF CARD 7	(INCHES)
9	AL((I), I=6,10	5F8.4	SAME AS CARD 8, AT SECOND FIVE RADII OF CARD 7	(INCHES)
10	A2 (I), I=1,5	5F8.4	SAME AS CARD 8, FOR ROTOR BLADE	(INCHES)
11	A2 (I), I=6,10	5F8.4	SAME AS CARD 9, FOR ROTOR BLADE	(INCHES)
12	AL ALI AL ϕ	F8.5 F8.5 F8.5	STATOR BLADE CAMBER LINE LENGTH, MEAN SECTION STATOR BLADE CAMBER LINE LENGTH, HUB SECTION STATOR BLADE CAMBER LINE LENGTH, TIP SECTION	SEE FIG. 6 (INCHES)
13	C CI C ϕ	F8.5 F8.5 F8.5	STATOR BLADE CHORD, MEAN SECTION STATOR BLADE CHORD, HUB SECTION STATOR BLADE CHORD, TIP SECTION	SEE FIG. 6 (INCHES)
14	E EI E ϕ	F8.5 F8.5 F8.5	STATOR BLADE CURVATURE, MEAN SECTION STATOR BLADE CURVATURE, HUB SECTION STATOR BLADE CURVATURE, TIP SECTION	SEE FIG. 6 (INCHES)
15	T TI T ϕ	F8.5 F8.5 F8.5	STATOR BLADE MAXIMUM THICKNESS, MEAN SECTION STATOR BLADE MAXIMUM THICKNESS, HUB SECTION STATOR BLADE MAXIMUM THICKNESS, TIP SECTION	SEE FIG. 6 (INCHES)
16	TE TEI TE ϕ	F8.5 F8.5 F8.5	STATOR BLADE T.E. PROJECTED THICKNESS, MEAN SECTION STATOR BLADE T.E. PROJECTED THICKNESS, HUB SECTION STATOR BLADE T.E. PROJECTED THICKNESS, TIP SECTION	SEE FIG. 6 (INCHES)
17	TN TNI TN ϕ	F8.5 F8.5 F8.5	STATOR BLADE T.E. NORMAL THICKNESS, MEAN SECTION STATOR BLADE T.E. NORMAL THICKNESS, HUB SECTION STATOR BLADE T.E. NORMAL THICKNESS, TIP SECTION	SEE FIG. 6 (INCHES)

TABLE II (2)

CARD	FORTRAN NAME	FORMAT	DESCRIPTION	COMMENTS
18-23		F8.5	SAME OF CARDS 12-17, FOR ROTOR BLADE	(INCHES)
24	RC (1) RC (5)	F8.4 F8.4	HUB RADIUS AT STATION 0 TIP RADIUS AT STATION 0	(INCHES)
25	RS (1) RS (5)	F8.4 F8.4	HUB RADIUS AT STATION 1 HUB RADIUS AT STATION 1	(INCHES)
26	RR (1) RR (5)	F8.4 F8.4	HUB RADIUS AT STATION 2 TIP RADIUS AT STATION 2	(INCHES) WITHOUT TIP CLEARANCE
27	TOL1 TOL2 TOL3 TOL4	F8.4 F8.4 F8.4 F8.4	REL. TOLERANCE IN OVERALL CONTINUITY REL. TOLERANCE IN STREAMLINE CONTINUITY REL. TOLERANCE IN PRESSURE RATIO REL. TOLERANCE IN VELOCITY DISTRIBUTION	
28	NSETS	I5	NUMBER OF SETS OF OPERATING CONDITIONS FOR WHICH SOLUTIONS HAVE TO BE FOUND	
(28+NSETS) + 29	AMC AMS PTP TTP AMR RPM PR VAL(3) VA2(3)	F8.4 F8.4 F8.4 F8.4 F8.4 F8.4 F8.4 F8.4 F8.4	MACH NUMBER AT STATION 0 MACH NUMBER AT STATION 1 TOTAL PRESSURE AT STATION 0 TOTAL TEMPERATURE AT STATION 0 MACH NUMBER AT STATION 2 SPEED OF REVOLUTION TOTAL-STATIC PRESSURE RATIO AXIAL VELOCITY AT MEAN STR. AT STATION 1 AXIAL VELOCITY AT MEAN STR. AT STATION 2	APPROXIMATE VALUE APPROXIMATE VALUE (PSI) ASSUMED TO BE CONSTANT (°R) ASSUMED TO BE CONSTANT APPROXIMATE VALUE (REVOLUTION/MINUTE) MASS AVERAGED QUANTITY APPROXIMATE VALUE APPROXIMATE VALUE

TABLE II (3)

PRINT INDEX	POSSIBLE VALUES	SPECIFICATIONS
IND	1	Intermediate results are printed in subroutines CHAN, STATØR, RØTØR2, FLØWR, SLINE
	≠ 1	No intermediate prints.
INZ	1	Solutions are printed for all calculated pressure ratios
	≠ 1	Solutions are printed only for specified pressure ratios
IWR	1	Intermediate results in loss coefficients subroutines are printed
	≠ 1	No intermediate prints

TABLE IV

CORRELATION INDEX	POSSIBLE VALUES	SPECIFICATIONS
ICOR	1	A&M [6] or D&C [7] loss correlation method is adopted, depending on IAI value.
	2	Balje [8] loss correlation method is adopted.
	3	I&C [9] loss correlation method is adopted.
	4	Traupel [10] loss correlation method is adopted.
IAI	0	If ICOR=1, A&M [6] loss correlation method is adopted.
	1	If ICOR=1, D&C [7] loss correlation method is adopted.
IAN	1	A&M [6] flow angles correlation is adopted.
	2	Traupel [10] flow angles correlation is adopted.
ICL	0	Rotor is not shrouded.
	1	Rotor is shrouded.

TABLE V (1)

CORRELATION INDEX	POSSIBLE VALUES	SPECIFICATIONS
IINC	0	Loss coefficients are function of flow angles
	1	Loss coefficients are function of blade inlet angles, for negative incidence
IC ϕ Z	1	Loss coefficient $\zeta = \zeta_0$ (see eq. (42))
	6	Loss coefficient $\zeta = \zeta(Y, \text{pressure ratio})$ (see eq. (41))
	8	Loss coefficient $\zeta = \zeta_{M=0.8}$ (see eq. (43))
IC ϕ N	1	Loss coefficient in continuity equation $\zeta^* = .5 \zeta_{TOT}$
	2	Loss coefficient in continuity equation $\zeta^* = \zeta_{PROF}$
	3	Loss coefficient in continuity equation $\zeta^* = \zeta_{TOT}$

TABLE V (2)

TABLE VI

EXAMPLE OF NORMAL OUTPUT DATA

R1	A1	R2	A2
10.500	0.5252	10.500	0.6250
11.000	0.5855	11.000	0.6410
11.500	0.6459	11.500	0.6570
12.000	0.7002	12.000	0.6730
12.500	0.7665	12.500	0.6880
13.000	0.8268	13.000	0.7030
13.500	0.8881	13.500	0.7170
14.000	0.9494	14.000	0.7390
14.500	1.0108	14.500	0.7400
15.000	1.0722	15.000	0.7480

NUMBER OF STATOR BLADES	=	40°
NUMBER OF ROTOR BLADES	=	76°
ROTOR TIP CLEARANCE	=	0.0300
AXIAL DISTANCE L	=	2.50
CURVATURE FACTOR K	=	5.00

BLADING GEOMETRY

	C	E	T	TE	TN	AL	R
STATOR	2.5400	20.0000	0.3300	0.0740	0.0250	2.8200	10.5000
	2.8300	20.0000	0.3300	0.0631	0.0250	2.9600	12.7500
	3.1500	20.0000	0.3600	0.0560	0.0250	3.3400	15.0000
ROTOR	2.1300	5.3000	0.3840	0.0460	0.0300	2.4600	10.5000
	2.1600	8.0000	0.3800	0.0435	0.0300	2.4700	12.7500
	2.1600	8.0000	0.2500	0.0384	0.0300	2.5000	15.0000

ALL DIMENSIONS INDICATED IN THIS TABLE ARE IN INCHES

GAS PROPERTIES	CORRELATION SYSTEM	ICOR	=	4	GAMMA	VISCOSITY (1) (LBM /SEC FT)	VISCOSITY (2) (LBM /SEC FT)
		IAI	=	0			
		IAN	=	2			
		ICDZ	=	1			
		INC	=	1			
	MOLECULAR MASS	ICL	=	0	1.400	0.130E-04	0.120E-04
		ICON	=	2			
		CP (BTU/LB F)	28.970				

SET NUMBER	PAGE NUMBER	RPM	TOTAL/STATIC PRESSURE RATIO	INLET TOTAL PRESSURE (PSI)	INLET TOTAL TEMPERATURE (DEG. R)
1	1	32620.0	1.500	44.000	550.00

STATOR EXIT SOLUTION

STREAM LINE	RADIAL POSITION (IN)	X=R/RM	RADIAL SHIFT (IN)	BLADE OPENING (IN)	Y=VA/VAM	BLADE EFFICIENCY	LOSS COEFFICIENT	ZETA* CONTINUITY	FLOW RATE FRACTION
1	10.500	0.802	0.0	0.5240	0.9172	0.9203	0.0797	0.0366	0.0
2	11.917	0.911	-0.1301	0.6920	0.9642	0.9215	0.0785	0.0352	0.2491
3	13.085	1.000	-0.1379	0.8286	1.0000	0.9224	0.0776	0.0342	0.5001
4	14.095	1.077	-0.0838	0.9569	1.0200	0.9229	0.0771	0.0331	0.7501
5	15.000	1.146	0.0	1.0733	1.0296	0.9233	0.0767	0.0324	1.0000

ABSOLUTE VELOCITY (FPS)

STREAM LINE	AXIAL COMPONENT	RADIAL COMPONENT	TANGENTIAL COMPONENT	OVERALL VELOCITY	AXIAL COMPONENT	RADIAL COMPONENT	TANGENTIAL COMPONENT	OVERALL VELOCITY	WHEEL VELOCITY
1	296.13	0.0	876.67	925.33	296.13	0.0	577.77	649.24	298.90
2	311.30	3.73	780.22	840.04	311.30	3.73	440.57	539.80	339.24
3	322.87	4.06	721.50	790.46	322.87	4.06	349.02	475.48	372.48
4	329.32	3.23	682.88	758.14	329.32	3.23	281.65	433.34	401.23
5	332.41	0.0	646.46	726.91	332.41	0.0	219.46	398.32	427.00

MACH NUMBER

STREAM LINE	FLOW ANGLE (DEG)		TEMPERATURE (DEG. R)		PRESSURE (PSI)		PRESSURE RATIO	
	ABSOLUTE	RELATIVE	TOTAL	STATIC	TOTAL	STATIC	TOT/TOT	TOT/STA
1	0.86	0.61	550.00	478.75	42.048	25.874	1.0464	1.7006
2	0.77	0.50	550.00	491.28	42.453	28.595	1.0364	1.5387
3	0.72	0.43	550.00	498.01	42.682	30.137	1.0314	1.4600
4	0.69	0.39	550.00	502.17	42.786	31.119	1.0284	1.4140
5	0.66	0.36	550.00	506.03	42.868	32.047	1.0257	1.3730

SET NUMBER	PAGE NUMBER	RPM	TOTAL/STATIC PRESSURE RATIO	INLET TOTAL PRESSURE (PSI)	INLET TOTAL TEMPERATURE (DEG. R)
1	2	3262.0	1.500	44.000	550.00

ROTOR EXIT SOLUTION

STREAM LINE	RADIAL POSITION	X=R/RM	RADIAL SHIFT	BLADE OPENING	Y=VA/VAM	BLADE EFFICIENCY	LOSS COEFFICIENT	ZETA* CONTINUITY	FLOW RATE FRACTION
1	10.500	0.811	0.0	0.6251	1.0222	0.8277	0.1723	0.0879	0.0
2	11.785	0.911	-0.0599	0.6662	1.0312	0.8562	0.1438	0.0641	0.2505
3	12.940	1.000	-0.0628	0.7006	1.0000	0.8729	0.1271	0.0502	0.5004
4	14.017	1.083	-0.0490	0.7274	0.9749	0.8846	0.1154	0.0419	0.7505
5	15.000	1.159	0.0	0.7479	0.9680	0.8924	0.1076	0.0363	1.0000

ABSOLUTE VELOCITY (FPS)				RELATIVE VELOCITY (FPS)			
STREAM LINE	AXIAL COMPONENT	RADIAL COMPONENT	TANGENTIAL COMPONENT	OVERALL VELOCITY	AXIAL COMPONENT	RADIAL COMPONENT	TANGENTIAL COMPONENT
1	328.25	0.0	-0.30	328.25	328.25	0.0	-299.20
2	331.14	3.97	-0.46	331.14	331.14	3.97	-335.93
3	321.07	4.04	10.51	321.07	321.07	4.04	-357.83
4	313.07	3.07	22.43	313.07	313.07	3.07	-376.59
5	310.86	0.0	25.66	310.86	310.86	0.0	-401.34

MACH NUMBER				TEMPERATURE (DEG. R)				PRESSURE (PSI)			
STREAM LINE	ABSOLUTE	RELATIVE	ABSOLUTE	RELATIVE	TOTAL	STATIC	TOTAL	STATIC	TOT/TOT	TOT/STA	
1	0.30	0.41	-0.05	-42.35	506.38	497.41	30.736	28.873	1.4315	1.5239	
2	0.30	0.43	-0.08	-43.42	505.92	496.90	31.002	28.109	1.4193	1.5116	
3	0.29	0.44	1.87	-48.10	505.89	497.45	31.223	28.432	1.4092	1.4950	
4	0.29	0.45	4.10	-50.27	505.88	497.77	31.352	29.626	1.4034	1.4852	
5	0.29	0.46	4.72	-52.24	505.88	497.79	31.437	29.711	1.3996	1.4810	

EQUIV/STATIC PRESSURE RATIO				INCIDENCE ANGLE (DEG)			
STREAM LINE	EQUIVALENT TEMPERATURE (DEG. R)	EQUIVALENT INLET PRESSURE (PSI)	BLADE INLET ANGLE (DEG)	BLADE STALL ANGLE (DEG)	INCIDENCE ANGLE (DEG)		
1	513.82	33.14	1.148	11.0	12.4		
2	515.41	33.82	1.162	11.8	7.5		
3	516.69	34.28	1.165	13.0	4.3		
4	517.73	34.63	1.169	15.2	2.5		
5	519.23	35.07	1.180	18.5	0.8		

SET NUMBER	PAGE NUMBER	RPM	TOTAL/STATIC PRESSURE RATIO	INLET TOTAL PRESSURE (PSI)	INLET TOTAL TEMPERATURE (DEG. R)
1	3	3262.0	1.500	44.000	550.00

OVERALL TURBINE CHARACTERISTICS

STREAM LINE	PRESSURE RATIO TOT/TOT	EFFICIENCY TOT/STA	TOT/TOT	HEAD COEFFICIENT	BLADE/JET SPEED RATIO	THEORETICAL DEGREE OF REACTION
1	1.4315	0.6994	0.8142	8.3899	0.3422	-0.2412
2	1.4193	0.7197	0.8418	6.3945	0.3925	-0.0405
3	1.4092	0.7384	0.8585	5.1708	0.4398	0.0558
4	1.4034	0.7505	0.8689	4.3874	0.4774	0.1182
5	1.3996	0.7558	0.8757	3.8475	0.5098	0.1842

MASS AVERAGED QUANTITIES

HORSE POWER = 1954.03 (HP)
MOMENT = 3146.15 (FT-LB)
FLOW RATE = 130.69 (LB/SEC)

REFERRED RPM = 3166.91 (HP)
REFERRED HORSE POWER = 633.79 (FT-LB)
REFERRED MOMENT = 1051.10 (LB/SEC)
REFERRED FLOW RATE = 44.97 (LB/SEC)

TOTAL/STATIC EFFICIENCY = 0.7341
TOTAL/TOTAL EFFICIENCY = 0.8535

TOTAL/STATIC PRESSURE RATIO = 1.4985
TOTAL/TOTAL PRESSURE RATIO = 1.4119

HEAD COEFFICIENT = 5.5178
BLADE/JET SPEED RATIO = 0.4257
THEORETICAL DEGREE OF REACTION = 0.0262
MACH NUMBER AT STATION 0 = 0.2162

APPENDIX


```

0018 ZTOR, O1OR, BETAI, ANM, TIR, TR, TOR, STALI, PTE, RS1, RS3, RS5, T2
0019 8LXX=1.025
0020 IF (ICL=7) XCL=1.035
0021 IF (ICL=7) XCL=7
0022 CALL CODEF12
0023 CALL CODEF12
0024 CALL CODEF12
0025 CALL CODEF12
0026 CALL CODEF12
0027 CALL CODEF12
0028 CALL CODEF12
0029 CALL CODEF12
0030 CALL CODEF12
0031 CALL CODEF12
0032 CALL CODEF12
0033 CALL CODEF12
0034 CALL CODEF12
0035 CALL CODEF12
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0040 CALL CODEF12
0041 CALL CODEF12
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0046 CALL CODEF12
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0050 CALL CODEF12
0051 CALL CODEF12
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0054 CALL CODEF12
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0070 CALL CODEF12
0071 CALL CODEF12
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0075 CALL CODEF12
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0077 CALL CODEF12
0078 CALL CODEF12
0079 CALL CODEF12
0080 CALL CODEF12
0081 CALL CODEF12
0082 CALL CODEF12
0083 CALL CODEF12
0084 CALL CODEF12
0085 CALL CODEF12
0086 CALL CODEF12
0087 CALL CODEF12
0088 CALL CODEF12
0089 CALL CODEF12
0090 CALL CODEF12
0091 CALL CODEF12
0092 CALL CODEF12
0093 CALL CODEF12
0094 CALL CODEF12
0095 CALL CODEF12
0096 CALL CODEF12
0097 CALL CODEF12
0098 CALL CODEF12
0099 CALL CODEF12
0100 CALL CODEF12

```

```

0088 IF(RS(3).LE.R1(I)) GOTO 3712
0089 CONTINUE
0090 CONTINUE
0091 I=1
0092 AL(I)=AL(I+1)-AL(I)*(RS(3)-R1(I))/(R1(I+1)-R1(I))
0093 IF(RR(1).LE.R2(I)) GOTO 3714
0094 CONTINUE
0095 CONTINUE
0096 I=1
0097 OR=A2(I)+(A2(I+1)-A2(I))*(RR(3)-R2(I))/(R2(I+1)-R2(I))
0098 H=RS(5)-RS(1)
0099 O=O.*RS(3)
0100 S=2.*3.1416*RS(3)/ZS
0101 SI=2.*3.1416*RS(1)/ZS
0102 SO=2.*3.1416*RS(5)/ZS
0103 SZ=2.*3.1416*RR(3)/ZR
0104 SK=2.*3.1416*RR(1)/ZR
0105 OM=2.*3.1416*RR(5)/ZR
0106 HR=RR(5)-RR(1)
0107 ANG1=O.

```

AINLEY METHOD , STATOR EXIT ANGLES AT DIFFERENT MACH NUMBERS

```

0108 CL= TIP
0109 CALL ANGIN(ANG1,ALFAM(1),S,C,O,E,TE,O.5,O.,H,O,IWR,ANG21)
0110 CALL ANGIN(ANG1,ALFAM(2),S,C,O,E,TE,O.7,O.,H,O,IWR,ANG21)
0111 CALL ANGIN(ANG1,ALFAM(3),S,C,O,E,TE,O.75,O.,H,O,IWR,ANG21)
0112 CALL ANGIN(ANG1,ALFAM(4),S,C,O,E,TE,O.8O,O.,H,O,IWR,ANG21)
0113 CALL ANGIN(ANG1,ALFAM(5),S,C,O,E,TE,1.O,O.,H,O,IWR,ANG21)
0114 ALFAM(1)=1.
0115 ALFAM(2)=2.
0116 ALFAM(3)=3.
0117 ALFAM(4)=4.
0118 ALFAM(5)=5.
0119 O1=O.
0120 O2=2.
0121 O3=3.
0122 O4=4.
0123 O5=5.
0124 CALL ANGIN(ANG1,ANG21,SI,CI,O1,EI,TEI,O.5,O.,H,O,IWR,ANG21)
0125 CALL ANGIN(ANG1,ANG21,SO,CO,OO,EO,TEO,O.5,O.,H,O,IWR,ANG21)
0126 DOLF(1)=O.
0127 DOLF(3)=O.
0128 DOLF(5)=-ANG2O-ALFAM(1)

```

ROTOR EXIT ANGLES

```

0129 O1R=2.*RR(1)
0130 O1O=2.*RO(5)
0131 ANI=BEY(3)
0132 CALL ANGIN(ANG1,BETAM(1),SZ,CR,OR,ER,TER,O.5,CL,HR,OZ,IWR,ANM)
0133 CALL ANGIN(ANG1,BETAM(2),SZ,CR,OR,ER,TER,O.7,CL,HR,OZ,IWR,ANG21)
0134 CALL ANGIN(ANG1,BETAM(3),SZ,CR,OR,ER,TER,O.75,CL,HR,OZ,IWR,ANG21)
0135 CALL ANGIN(ANG1,BETAM(4),SZ,CR,OR,ER,TER,O.8O,CL,HR,OZ,IWR,ANG21)
0136 CALL ANGIN(ANG1,BETAM(5),SZ,CR,OR,ER,TER,1.OO,CL,HR,OZ,IWR,ANG21)
0137 ANI=BETA(1)
0138 CALL ANGIN(ANG1,BETAO(1),SIR,CIR,OIR,EIR,TEIR,O.5,O.,HR,OZ,IWR,AN)
0139 ANI=BETA(5)
0140 CALL ANGIN(ANG1,BETA(5),SOR,COR,DOO,EOO,TEOO,O.5,O.,HR,OZ,IWR,AN)
0141 DBET(1)=BETAI-BETAM(1)
0142 DBET(3)=O.
0143 ANI=BETAO(5)
0144 CALL ANGIN(ANG1,BETAC(5),SOR,COR,DOO,EOO,TEOO,O.5,CL,HR,OZ,IWR,AN)
0145 DBET(5)=BETAC-BETAM(1)
0146 IF(TANG.EQ.1) GOTO 3377

```

ANGLES COMPUTATION WITH TRAUPEL METHOD

```

00 3370 I=15
0147 ALFAM(1)=ANGTRAITN(S,O)
0148 ANG21=ANGTRAITN(SI,OI)
0149 ANG2O=ANGTRAITN(SO,OO)

```



```

FORTRAN IV G LEVEL 18      MAIN      OATE = 70236      18/50/51

0216 1RR(3),COR,EOR,TOR,TEOR,TNOR,ALOR,RR(5)
0217 WRITE(6,78)
0218 78 FORMAT(/40X,52HALL DIMENSIONS INOICATED IN THIS TABLE ARE IN INCHE
0219 1S/)
0220 WRITE(6,79)ICOR,IAI,IAN,IC02,IINC,ICL,ICON
0221 79 FORMAT(////,40X,27HCORRELATION SYSTEM ICOR =,15/61X,6HIAI =,15/
161X,6HIAN =,15/61X,6HIC02 =,15/61X,6HIINC =,15/61X,6HICL =,15/6
21X,6HICON =,15/)
0222 81 WRITE(6,81)CP,EMME,GAM,VIS2,VIS3
0223 81 FORMAT(/20X,91HGAS,PROPERTIES CP MOLECULAR MASS
31X,10M /SEC F1,4X,13H(L8M /SEC F1)/36X,F9.3,5X,F10.3,9X,F10.3,2E15
31X,1)
0224 4 FORMAT(815)
0225 400 J=1,NSETS
0226 400 J=1,NSETS
0227 400 J=1,NSETS
0228 400 J=1,NSETS
0229 400 J=1,NSETS
0230 400 J=1,NSETS
0231 400 J=1,NSETS
0232 400 J=1,NSETS
0233 400 J=1,NSETS
0234 400 J=1,NSETS
0235 400 J=1,NSETS
0236 400 J=1,NSETS
0237 400 J=1,NSETS
0238 400 J=1,NSETS
0239 400 J=1,NSETS
0240 400 J=1,NSETS
0241 400 J=1,NSETS
0242 400 J=1,NSETS
0243 400 J=1,NSETS
0244 400 J=1,NSETS
0245 400 J=1,NSETS
0246 400 J=1,NSETS
0247 400 J=1,NSETS
0248 400 J=1,NSETS
0249 400 J=1,NSETS
0250 400 J=1,NSETS
0251 400 J=1,NSETS
0252 400 J=1,NSETS
0253 400 J=1,NSETS
0254 400 J=1,NSETS
0255 400 J=1,NSETS
0256 400 J=1,NSETS
0257 400 J=1,NSETS
0258 400 J=1,NSETS
0259 400 J=1,NSETS
0260 400 J=1,NSETS
0261 400 J=1,NSETS
0262 400 J=1,NSETS
0263 400 J=1,NSETS
0264 400 J=1,NSETS
0265 400 J=1,NSETS
0266 400 J=1,NSETS
0267 400 J=1,NSETS
0268 400 J=1,NSETS
0269 400 J=1,NSETS
0270 400 J=1,NSETS
0271 400 J=1,NSETS
0272 400 J=1,NSETS
0273 400 J=1,NSETS
0274 400 J=1,NSETS
0275 400 J=1,NSETS
0276 400 J=1,NSETS
0277 400 J=1,NSETS
0278 400 J=1,NSETS

```


18/50/51

DATE = 70236

CHAN

FORTRAN IV G LEVEL 18

```

0001 SUBROUTINE CHAN (TTO,AMC,PTO,RC,WLBM,WCHAN,MPERO)
0002 COMMON /GAS/ CP,GAM,EMME,ERRE,EXPI,EXP2,VIS2,VIS3
0003 DIMENSION KC(10),MPERO(10)
0004 C=5.0D-11**GAM*(10)**AMC
0005 VC=5.0D-11**GAM*(2)**AMC
0006 PC=5.0D-11**GAM*(2)**AMC
0007 RC=PC/ERRE/VC
0008 AREA=3.1416*(RC(5)**2-RC(1)**2)
0009 WLBM=RHO*AREA*VC
0010 WCHAN=WLBM/(PTO*SORT(G/ERRE/TTO))
0011 MPERO(1)=0
0012 MPERO(2)=25
0013 MPERO(3)=5
0014 MPERO(4)=75
0015 MPERO(5)=1
0016 RETURN
0017 END
0018

```

```

0001 SUBROUTINE STATOR (ALFA1,X,TTO,PTO,AM,T,P,V1,V2,VAL,S11,S12,Y,S,OSOX,
0002 1,VAL,S11,S12,Y,S,OSOX,
0003 1,VAL,S11,S12,Y,S,OSOX,
0004 1,VAL,S11,S12,Y,S,OSOX,
0005 1,VAL,S11,S12,Y,S,OSOX,
0006 1,VAL,S11,S12,Y,S,OSOX,
0007 1,VAL,S11,S12,Y,S,OSOX,
0008 1,VAL,S11,S12,Y,S,OSOX,
0009 1,VAL,S11,S12,Y,S,OSOX,
0010 1,VAL,S11,S12,Y,S,OSOX,
0011 1,VAL,S11,S12,Y,S,OSOX,
0012 1,VAL,S11,S12,Y,S,OSOX,
0013 1,VAL,S11,S12,Y,S,OSOX,
0014 1,VAL,S11,S12,Y,S,OSOX,
0015 1,VAL,S11,S12,Y,S,OSOX,
0016 1,VAL,S11,S12,Y,S,OSOX,
0017 1,VAL,S11,S12,Y,S,OSOX,
0018 1,VAL,S11,S12,Y,S,OSOX,
0019 1,VAL,S11,S12,Y,S,OSOX,
0020 1,VAL,S11,S12,Y,S,OSOX,
0021 1,VAL,S11,S12,Y,S,OSOX,
0022 1,VAL,S11,S12,Y,S,OSOX,
0023 1,VAL,S11,S12,Y,S,OSOX,
0024 1,VAL,S11,S12,Y,S,OSOX,
0025 1,VAL,S11,S12,Y,S,OSOX,
0026 1,VAL,S11,S12,Y,S,OSOX,
0027 1,VAL,S11,S12,Y,S,OSOX,
0028 1,VAL,S11,S12,Y,S,OSOX,
0029 1,VAL,S11,S12,Y,S,OSOX,
0030 1,VAL,S11,S12,Y,S,OSOX,
0031 1,VAL,S11,S12,Y,S,OSOX,
0032 1,VAL,S11,S12,Y,S,OSOX,
0033 1,VAL,S11,S12,Y,S,OSOX,
0034 1,VAL,S11,S12,Y,S,OSOX,
0035 1,VAL,S11,S12,Y,S,OSOX,
0036 1,VAL,S11,S12,Y,S,OSOX,
0037 1,VAL,S11,S12,Y,S,OSOX,
0038 1,VAL,S11,S12,Y,S,OSOX,
0039 1,VAL,S11,S12,Y,S,OSOX,
0040 1,VAL,S11,S12,Y,S,OSOX,
0041 1,VAL,S11,S12,Y,S,OSOX,
0042 1,VAL,S11,S12,Y,S,OSOX,
0043 1,VAL,S11,S12,Y,S,OSOX,
0044 1,VAL,S11,S12,Y,S,OSOX,
0045 1,VAL,S11,S12,Y,S,OSOX,
0046 1,VAL,S11,S12,Y,S,OSOX,
0047 1,VAL,S11,S12,Y,S,OSOX,
0048 1,VAL,S11,S12,Y,S,OSOX,
0049 1,VAL,S11,S12,Y,S,OSOX,
0050 1,VAL,S11,S12,Y,S,OSOX,
0051 1,VAL,S11,S12,Y,S,OSOX,
0052 1,VAL,S11,S12,Y,S,OSOX,
0053 1,VAL,S11,S12,Y,S,OSOX,
0054 1,VAL,S11,S12,Y,S,OSOX,
0055 1,VAL,S11,S12,Y,S,OSOX,
0056 1,VAL,S11,S12,Y,S,OSOX,
0057 1,VAL,S11,S12,Y,S,OSOX,
0058 1,VAL,S11,S12,Y,S,OSOX,
0059 1,VAL,S11,S12,Y,S,OSOX,
0060 1,VAL,S11,S12,Y,S,OSOX,
0061 1,VAL,S11,S12,Y,S,OSOX,
0062 1,VAL,S11,S12,Y,S,OSOX,
0063 1,VAL,S11,S12,Y,S,OSOX,
0064 1,VAL,S11,S12,Y,S,OSOX,
0065 1,VAL,S11,S12,Y,S,OSOX,

```



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FORTRAN IV G LEVEL 18      ALOS1      DATE = 70236      18/50/51
0065  ANUM=1-(10-XE)*(10/FI+XE/F2+XE**2/F3+XE**3/F4+XE**4/F5+XE**5/F6)
0066  ADEN=1-ANUM-(10-XE)*(10/G1+XE/G2+XE**2/G3+XE**3/G4+XE**4/G5+XE**5/
1061  1061)
0067  XX=ANUM/ADEN
0068  BAL2=BAL1-ALFAL(MACC)
0069  BAL3=PI(MACC)/11(MACC)/ERRE*12.
0070  BAL4=RPMM*3.14159/30*RS(MACC)/12.
0071  IF(MACO-31.202,2023,2024)
0072  CALL BALJE(BAL1,BAL2,AL1,C1,SI,TEI,BAL3,VIS2,BAL4,BAL5,XX,0.,IWR,0
1062  1062)
0073  BAL6=BAL7
0074  GO TO 2025
0075  CALL BALJE(BAL1,BAL2,AL1,C1,S,TE,BAL3,VIS2,BAL4,BAL5,XX,0.,IWR,0
1063  1063)
0076  BAL6=BA26
0077  BAL7=BAL7
0078  GO TO 2025
0079  CALL BALJE(BAL1,BAL2,AL1,BAL3,ALO,CO,SO,TEO,BAL3,VIS2,BAL4,BAL5,XX,0.,IWR,0
1064  1064)
0080  BAL6=BA26
0081  BAL7=BAL7
0082  GO TO 2025
0083  ZETAS(MACC) = 8AL6 +BAL7+ BAL8
0084  CONTINUE
0085  IF(ICO2.LV.5) GOTO 2026
0086  DO 2027,MACC=1,52
0087  XY=ZETAS(MACC)/(1-ZETAS(MACC))
1065  1065)
0088  ZETAS(MACC)=((10+XY)/(10+XY**PI(MACC)/PTO1)) **EXP2-1.0/
1066  1066)
0089  IF(ICO2.EQ.8) ZETAS(MACC)=((10+XY)/(10+XY*BESPI))**EXP2-1.0/
1067  1067)
0090  CONTINUE
0091  CONTINUE
0092  IF(ICO2.NE.3) GOTO 31
0093  DO 32,MACC=1,52
0094  ZETAS(MACC)=1.52
0095  IF(ICO2.NE.4) GOTO 33
0096  IF(ICO2.EQ.4) ZETAS(3)=TR16
0097  IF(ICO2.EQ.4) ZETAS(5)=TR36
0098  IF(ICO2.EQ.4) ZETAS(3)=BA26
0099  IF(ICO2.EQ.4) ZETAS(5)=BA26
0100  IF(ICO2.EQ.4) ZETAS(3)=Z11
0101  IF(ICO2.EQ.4) ZETAS(5)=Z11/(10+Z11)
0102  IF(ICO2.EQ.4) ZETAS(3)=Z11/(10+Z11)
0103  IF(ICO2.EQ.4) ZETAS(5)=Z11/(10+Z11)
0104  IF(ICO2.NE.4) GOTO 34
0105  DO 35,MACC=1,52
0106  ZETAS(1)=ZETAS(1)
0107  ZETAS(1)=ZETAS(1)
0108  CONTINUE
0109  RETURN
0110  END

```

```

FORTRAN IV G LEVEL 18      ALOS2      DATE = 70236      18/50/51
0001      SUBROUTINE ALOS2(ZETAR,ZETAPR)
0002      DIMENSION ALF2(10),V1(10),RS(10),P1(10),T1(10),U (10),AMS1(10),
0003      1ZETAS(10),ZETAP2(10),W2(10),RR(10),P2(10),T2(10),U2(10),AMR2(10),
0004      2BETA2(10),ZETAPR(10),STAL1(10),PTET(10),BETA1(10),TTE(10)
0005      COMMON /ABA/ BA17,BLEX
0006      COMMON /GAS/ CO,GAM,EMME,ERRE,EXP1,EXP2,VIS2,VIS3
0007      COMMON /CSS/ CA,CA1
0008      COMMON /ALI/ ALFA1,ALFA2
0009      COMMON /XTE/ XTET,XTET2
0010      COMMON /AUS/ IAU,IUA,IUO,INZ,INR
0011      COMMON /ARE/ IARE,XELE
0012      COMMON /COZ/ ICOR,ICOZ,IINC,IAI,ICL,IAN,ICON
0013      COMMON /ALZ/ ALZ1,BETA2
0014      COMMON /ALZ/ ALZ1,BETA2
0015      COMMON /ALZ/ ALZ1,BETA2
0016      COMMON /ALZ/ ALZ1,BETA2
0017      COMMON /ALZ/ ALZ1,BETA2
0018      COMMON /ALZ/ ALZ1,BETA2
0019      COMMON /ALZ/ ALZ1,BETA2
0020      COMMON /ALZ/ ALZ1,BETA2
0021      COMMON /ALZ/ ALZ1,BETA2
0022      COMMON /ALZ/ ALZ1,BETA2
0023      COMMON /ALZ/ ALZ1,BETA2
0024      COMMON /ALZ/ ALZ1,BETA2
0025      COMMON /ALZ/ ALZ1,BETA2
0026      COMMON /ALZ/ ALZ1,BETA2
0027      COMMON /ALZ/ ALZ1,BETA2
0028      COMMON /ALZ/ ALZ1,BETA2
0029      COMMON /ALZ/ ALZ1,BETA2
0030      COMMON /ALZ/ ALZ1,BETA2
0031      COMMON /ALZ/ ALZ1,BETA2
0032      COMMON /ALZ/ ALZ1,BETA2
0033      COMMON /ALZ/ ALZ1,BETA2
0034      COMMON /ALZ/ ALZ1,BETA2
0035      COMMON /ALZ/ ALZ1,BETA2
0036      COMMON /ALZ/ ALZ1,BETA2
0037      COMMON /ALZ/ ALZ1,BETA2
0038      COMMON /ALZ/ ALZ1,BETA2
0039      COMMON /ALZ/ ALZ1,BETA2
0040      COMMON /ALZ/ ALZ1,BETA2
0041      COMMON /ALZ/ ALZ1,BETA2
0042      COMMON /ALZ/ ALZ1,BETA2
0043      COMMON /ALZ/ ALZ1,BETA2
0044      COMMON /ALZ/ ALZ1,BETA2
0045      COMMON /ALZ/ ALZ1,BETA2
0046      COMMON /ALZ/ ALZ1,BETA2
0047      COMMON /ALZ/ ALZ1,BETA2
0048      COMMON /ALZ/ ALZ1,BETA2
0049      COMMON /ALZ/ ALZ1,BETA2
0050      COMMON /ALZ/ ALZ1,BETA2
0051      COMMON /ALZ/ ALZ1,BETA2
0052      COMMON /ALZ/ ALZ1,BETA2
0053      COMMON /ALZ/ ALZ1,BETA2
0054      COMMON /ALZ/ ALZ1,BETA2
0055      COMMON /ALZ/ ALZ1,BETA2
0056      COMMON /ALZ/ ALZ1,BETA2
0057      COMMON /ALZ/ ALZ1,BETA2
0058      COMMON /ALZ/ ALZ1,BETA2
0059      COMMON /ALZ/ ALZ1,BETA2
0060      COMMON /ALZ/ ALZ1,BETA2

```


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ALOS2

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0128
END


```

0001 SUBROUTINE ANGAIN(ANG1,ANG2,S,C,O,E,TE,AM,CL,H,D,IWR,ANG21)
0002 COMMON /AUS/ XCL
0003 ANGOL=3.1415/180
0004 AO= ARCCS( O/S)
0005 A1= -AO
0006 AO=+10*ANGOL-1.146*AO
0007 AO=AO-4*(S/E)*ANGOL
0008 ANG2= (-AO+5*ANGOL/20*(10*SIN(20*3.1415*(AM-0.75))))
0009 IF (IWR.EQ.1) WRITE(6,1) ANG2,AO,ANG1
0010 1 FORMAT(175X, 7H ALFA1=,3E15.4)
0011 ANG21=ANG2
0012 IF (CL.LE.0) GO TO 11
0013 IF (CL.EQ.0) GO TO 11
0014 AO =ATAN( (10-X*CL/H)*COS(ANG1)/COS(ANG2)) *TAN(ANG2)+X*CL/H*
0015 1 COS(ANG1)/COS(ANG2)* TAN(ANG1)
0016 A1= -ARCCS(O/S*(10-CL/H)+ CL/H + CL/D)
0017 ANG2= (-AO+(-A1+AO)/20*(10*SIN(20*3.1415*(AM-0.75))))
0018 IF (AM.LT.0.5) ANG2=AO
0019 IF (IWR.EQ.1) WRITE(6,1) ANG2,AO,ANG1
0020 11 RETURN
0021 END

```

FORTRAN IV G LEVEL 18	ANGTRA	DATE = 70236	18/50/51	PAGE 0001
0001	FUNCTION ANGTRA(T,S,A)			
0002	FK=SQRT(1.0-T/S#(2.0*A/S+T/S))			
0003	ANGTRA=ARCOS(A/S/FK)			
0004	RETURN			
0005	END			

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COEFFI

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```

0001 SUBROUTINE COEFFI
0002 DIMENSION Z(9,7),ORDIN(9),RX(9),RH(9),IR(9)
0003 COMMON /COE/ COEF(8,7),ASCIS(9),C(9),ALFA(7)
0004 NP=9
0005 READ(5,61) Z
0006 61 FORMAT(9I10,3I1)
0007 62 FORMAT(9F5,2)
0008 DO 1 K=1,7
0009 DO 2 N(K)=Z(K,1)
0010 2 ORDIN(N(K))=K,1
0011 IF(I=5) M=7
0012 IF(I=NE=5) M=3
0013 IF(I=EQ=3) M=2
0014 CALL CHBFT(ASCIS,ORDIN,NP,C,M,RX,RH,IR)
0015 LM=M+1
0016 DO 5 L=1,LM
0017 5 COEF(L,N)=C(L)
0018 CONTINUE
0019 1 READ(5,65)ALFA
0020 65 FORMAT(7F5,C)
0021 RETURN
0022 END
0023

```

```

0001 SUBROUTINE COEFF2
0002 DIMENSION Z(9,7),ORDIN(9),RX(9),RH(9),IR(9),
0003 CCMCN /CO2/ COEF(8,7),ASCIS(8),C(9),ALFA(6)
0004 NP=8
0005 DO 10 I=1,6
0006   READ(5,61) ( Z(K,I),K=1,NP)
0007   CONTINUE
0008   READ(7,62) ( ASCIS(K),K=1,NP)
0009   61 FORMAT(6F5.2)
0010   62 FORMAT(6F5.2)
0011   DO 1 I=1,6
0012     DO 2 K=1,NP
0013       ORDIN(K)=Z(K,I)
0014     M=4
0015     IF( I.EQ.1) M=3
0016     CALL CHBFT(ASCIS,ORDIN,NP,C,M,RX,RH,IR)
0017     LM= M+1
0018     DO 5 L=1,LM
0019       COEF(L,I)=C(L)
0020     5 CONTINUE
0021     READ(7,65) ALFA
0022     65 FORMAT(7F5.0)
0023     RETURN
0024   END

```

```

0001      SUBROUTINE CALYAO (SOLIO,ALFA,RISUL)
0002      USE CALYAO AFTER SUBROUTINE COEFFI
0003      COMMON/COE/COEF(8,7),ASCIS(9),C(9),ALFA(7)
0004      DIMENSION YNU(7),RX(6),RM(6),IR(6)
0005      DIMENSION YZ( 5),XZ(5)
0006      DO 1 I=6,5) M=7
0007      IF(I.EQ.5) M=3
0008      IF(I.EQ.3) M=2
0009      M1= M+1
0010      DO 2 K=1,M1
0011      2 C(K)= COEF(K,I)
0012      1 YNU(I)=YC(SOLIO,C,M)
0013      IF( ALFA(1).GT. ALFA(7)) GOTO 5
0014      IF( ALFA(1).GT. ALFA(3)) GOTO 3
0015      IF( ALFA(1).GT. ALFA(1)) GOTO 4
0016      IF( ALFA(1).GT. ALFA(2)) GOTO 6
0017      RISUL= YNU(2)+ ( YNU(3)-YNU(2))/{ ALFA(3)-ALFA(2)}*( ALFA1-ALFA(2)
0018      1) GOTO 1000
0019      5 RETURN
0020      RISUL= YNU(7)
0021      CONTINUE
0022      M=2
0023      MU=0
0024      DO 21 I=3,7
0025      MU= MU+1
0026      XZ(MU)= ALFA(1)
0027      YZ(MU)= YNU(1)
0028      CALL CH8FT( XZ,YZ, MU, C,M,RX,RM,IR)
0029      RISUL= YNU(1)
0030      RETURN
0031      4 RETURN
0032      6 RISUL= YNU(1)+ ( YNU(2)-YNU(1))/{ ALFA(2)-ALFA(1)}*( ALFA1- AL
0033      1FA(1))
0034      1000 RETURN
0035      ENO

```

```

0001      SUBROUTINE CALYAZ( SOLID,ALFA1,RISUL)
C
C      USE CALYAZ AFTER SUBROUTINE COEFF2
COMMON /CO2/ COEF(8,7),ASCIS(8),C(9),ALFA(6)
DIMENSION YNU(7),RX(6),RM(6),IR(6)
DO 1 I=1,6
  M=4
  IF( I.EQ.1) M=3
  M1=2+I-M1
  2 C(K)=COEF(I,I)
  1 YNU(I)=VC(SOLID,C,M)
  IF( ALFA1.LT. ALFA(1)) GOTO 16
  IF( ALFA1.GE. ALFA(4)) GOTO 10
  M=2
  CALL CH8FT( ALFA, YNU, 4,C,M, RX, RM, IR)
  RISUL=VC( ALFA1,C,M)
  GOTO 1000
10  CONTINUE
  1 IF( ALFA1.LE. ALFA(5)) RISUL= YNU(4)*(YNU(5)-YNU(4))/(ALFA(5)-
    ALFA(4))* (ALFA1- ALFA(4))
  IF( ALFA1.LC. ALFA(5)) GOTO 1000
  1 IF( ALFA1.EQ. ALFA(6)) RISUL= YNU(5)*(YNU(6)-YNU(5))/(ALFA(6)-
    ALFA(5))* (ALFA1- ALFA(6))
  GOTO 15
1000 RETURN
15  CONTINUE
16  RETURN
16  CONTINUE
16  RISUL= YNU(1)
  RETURN
  END

```



```

0001 FUNCTION FIG6(X)
0002 IF(X.LE.-3.0) FIG6=-1.0*1.92*X
0003 IF(X.LE.-2.0) FIG6=-1.0*1.92*X
0004 IF(X.LE.-1.0) FIG6=-1.0*1.92*X
0005 IF(X.LE.0.0) FIG6=-1.0*1.92*X
0006 IF(X.GT.0.0) FIG6=-1.0*1.92*X
0007 IF(X.GT.0.0) FIG6=-1.0*1.92*X
0008 IF(X.GT.0.0) FIG6=-1.0*1.92*X
0009 IF(X.GT.0.0) FIG6=-1.0*1.92*X
0010 IF(X.GT.0.0) FIG6=-1.0*1.92*X
0011 IF(X.GT.0.0) FIG6=-1.0*1.92*X
0012 RETURN

```

```

0001 SUBROUTINE AINLOS(ANG1,ANG2,S,C,T,O11,O10,O21,O20,CL,AH,IWR,YPO,
0002 1YSCL,YTOT,AING,ASTA,8,TN)
0003 COMMON /COZ/ICCR,ICDZ,ITNC,IAI,ICL,IAN,ICON
0004 COMMON /ARE/ REE
0005 C=1/8*(11+O.5251E-01
0006 CIG8(12)=O.10744E+00
0007 CIG8(13)=O.30641E+00
0008 CIG8(14)=O.32341E+01
0009 CIG8(15)=O.32341E+01
0010 ANZ2= ANG1

      LOSSES WITH AINLEY METHOD, AT ZERO INICIOENCE
      ANG2 = WITHOUT CLEARANCE - RADIANTS
      AING,S,C,T ARE GIVEN ACCORDING WITH AINLEY NOMENCLATURE
      IN MAIN PROGRAM COEFF1,COEFF2 MUST BE CALLED BEFORE AINLOS

0011 OCG1=AING*180./3.1415
0012 OCG2=AING*180./3.1415
0013 OCG3=AING*180./3.1415
0014 A1=3.1415*(O1C*O10-O21*O21)/4.*COS(AING)
0015 A2=3.1415*(O20*O20-O21*O21)/4.*COS(ANG2)
0016 ASC1=1/2*(IAN2/AN1)*2/(1.+(O11+O21)/(O10+O20))
0017 Z=OEG2
0018 CALL CALYAO(SOL,Z,Y80)
0019 CALL CALYAZ(SOL,Z,Y81)
0020 TCC=T/C
0021 I(TCC.GT.25) TCC=25
0022 I(TCC.LT.0.25) TCC=0.25
0023 YPO=(Y80+(DEG1/OEG2)*(OEG1/DEG2)*(Y81-Y80))*(5.*TCC)**(-AING/ANG

0024 12 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0025 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0026 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0027 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0028 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0029 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0030 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0031 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0032 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0033 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0034 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0035 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0036 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0037 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0038 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0039 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0040 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0041 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0042 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0043 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0044 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0045 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0046 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0047 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0048 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0049 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0050 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0051 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0052 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0053 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0054 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0055 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0056 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0057 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO
0058 1 I(IWR.EQ.1)WRITE(6,1) Y80,Y81,YPO

```

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AINLOS

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0059
0060
0061
ANG1= ANZZ
RETURN
END

```

0001 SUBROUTINE TRAUPI
0002 COMMON /TRA/, XPC1(5,8), XPC2(6,8), ALF1(8), ALF01(5), ALF02(6),
0003 DO1(1,8), Y1(10), XC(8), RX(30), RY(30), IR(30), Z(6), CI(4,8), C2(4,8),
0004 DO1(1,8), XPC1(J,I), J=1,5)
0005 READ(5,2)(XPC2(J,I), J=1,6)
0006 RFORMAT(5E7,4)
0007 2 FFORMAT(6E7,4)
0008 1 CONTINUE
0009 READ(5,3)(ALF1(I), I=1,8)
0010 READ(5,4)(ALF01(I), I=1,5)
0011 READ(5,5)(ALF02(I), I=1,6)
0012 3 FFORMAT(3E5,0)
0013 4 FFORMAT(3E5,0)
0014 5 FFORMAT(1E5,0)
0015 6 I=1,8
0016 7 I=1,5
0017 8 J=1,5
0018 9 Y(J)=XPC1(J,I)
0019 10 I=1,8
0020 11 J=1,6
0021 12 Y(J)=XPC2(J,I)
0022 13 CALL CH8FT(ALF01,Y,C,3,RX,RH,IR)
0023 14 CALL CH8FT(ALF02,Y,L,Z,2,RX,RH,IR)
0024 15 DO1(J,I)=1,4
0025 16 C(J,I)=Z(J)
0026 17 CONTINUE
0027 18 RETURN
0028 19 ENO
0029

```

```

0001 FUNCTION XPO(ANG1,ANG2)
0002 COMMON TRA,XPO1(8),XPO2(6,8),ALF1(8),ALF01(5),ALF02(6)
0003 1Y(10),Y(10),C(6),RX(30),R(30),I(30),Z(6),C1(4,8),C2(4,8)
0004 IF(ANG2=80.1)2,3
0005 1 CCNTINUE
0006 DO 4 I=1,8
0007 DO 5 J=1,4
0008 5 C(J)=C1(J,I)
0009 4 Y(I)=YC(ANG2,C,3)
0010 GO TO 10
0011 2 CCNTINUE
0012 DO 6 I=XPO1(8)
0013 6 Y(I)=XPO1(5,I)
0014 GO TO 10
0015 3 CCNTINUE
0016 DO 7 I=1,8
0017 DO 8 J=1,3
0018 8 C(J)=C2(J,I)
0019 7 Y(I)=YC(ANG2,C,2)
0020 GO TO 10
0021 10 CCNTINUE
0022 DO 11 I=1,7
0023 IF(ANG1-GE,ALF1(I),AND,ANG1-LE,ALF1(I+1)) GO TO 100
0024 IF(ANG1-GE,ALF1(I),GO TO 101
0025 IF(ANG1-GE,ALF1(8))GO TO 102
0026 11 CCNTINUE
0027 100 XPO=Y(1)+(Y(I+1)-Y(I))/(ALF1(I+1)-ALF1(I))*(ANG1-ALF1(I))/2.)*
0028 IF(ANG2-LT,40.) XPO=0.09-(XPO1(I,I+XPO1(I,I+1))/2.)*
0029 1(ANG2-20.)/20.
0030 RETURN
0031 XPO=Y(1)
0032 XPO=Y(8)
0033 RETURN
0034 END

```

[illegible]

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CID

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```

0001 SUBROUTINE CID(ANG1,T,DEL,CSID,PSID,PSIF,HM,DM)
0002 DIMENSION X(7),Y(7),Z(7),Y2(7)
0003 DATA OEL,T,SIN(ANG1),Y1(7),Y2(7)
0004 DATA X(7),Z(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0005 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0006 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0007 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0008 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0009 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0010 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0011 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0012 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0013 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0014 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0015 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0016 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0017 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0018 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0019 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)
0020 DATA Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7),Y(7),Y2(7)

```

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FORTTRAN IV G LEVEL 18 CSIW
0001 FUNCTION CSIW(XPO,CSIP,I,ANG1,AH)
0002 CSIW=XPO*CSIP+SIN(ANG1)/AH
0003 RETURN
0004 END

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CSIR

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```

0001 FUNCTION CSIR( S, AH, V1, ANGI, UM, XP)
0002 SL=SAH
0003 IF(SL<0.4) XL=XP*0.65/.4*SL
0004 IF(SL<0.4) AND(SL<0.8) XL=XP*(0.65/0.4*(SL-0.4))
0005 IF(SL<0.4) AND(SL<0.8) XL=XP*(1.1+0.40/0.7*(SL-0.8))
0006 IF(SL<0.4) AND(SL<0.8) XL=XP*(1.5+0.6/1.0*(SL-1.5))
0007 ASC=1*SL*ANGI/UM
0008 XRD=0.025+0.015*(0.66*ASC*ASC
0009 IF(SL<0.4) XRD=0.0275
0010 IF(SL<0.4) XRD=0.0475
0011 RETURN
0012 END
0013

```

```

0001 FUNCTION ALEAK (DELBET,DM,AL,CLE,ALFAL)
0002   CI=0.82-0.075*CELBET
0003   ALEAK=CI*
0004     (DM+AL)*CLE/DM/AL/COS(ALFAL)
0005 RETURN
      END

```

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```

0001 SUBROUTINE TRAP2(ANG1,ANGO,V1,TO,EMME,GAM ,T,DEZ,HM,DM,CSIP,S,UM
0002 1,CCL,RPRD,R2,R3,YCL,RTOT)
0003   R=XP0(ANG1,ANGO)
0004   R1=CS1M(V1,TO,EMME,GAM)
0005   ANGZ=ANG1*3.1415/180.
0006   CALL CIDIANG2(T,DEZ,CSID,PSID,PSIF,HM,DM)
0007   R2=CS1M(R,CSIP,T,ANGZ,HM,CSIP)
0008   R3=CS1M(S,HM,V1,ANGZ,UM,CSIP)
0009   RTOT=R2+R3+R1*CSID+PSIF+PSID
0010   T=CCL*1E-6) COT 0.000
0011   DEL=1.5708-ANGZ
0012   ALF1=1.5708-ANGZ
0013   YCL=ALF1*DEL*DM,HM,CL ,ALF1)
0014   RTOT=RTOT+R2+R3
0015   1000 RETURN
0016   END
0017

```

```

0001      FUNCTION ZAILOS(BETIN,BETEX,VIN,VEX)
0002      BETIN,BETEX=ANGLES WITH AXIAL DIRECTION, LIKE IN AINLEY
0003      ZAILOS=(TAN(BETIN)-TAN(BETEX))/(.05+.8*CO{8ETEX})
0004      IF{(VIN/VEX).LE.06) ZAILOS=ZAILOS*(.03+.1573*(VIN/VEX)**3.6)
0005      IF{(VIN/VEX).GT.06) ZAILOS=ZAILOS*(.055+.15*(VIN/VEX-.06))
0006      ZAILOS= ZAILOS/ (1.+ ZAILOS)
0007      RETURN
      END

```



```

0001 SUBROUTINE PARAB (V,I,J,K,A,B,C,VAR1,VAR2,VAR3)
0002 DIMENSION V(10)
0003 COMMON INO
0004 VAR12=VAR1**2
0005 VAR22=VAR2**2
0006 VAR32=VAR3**2
0007 D=VAR2*VAR32-VAR3*VAR22+VAR1*(VAR22-VAR32)+VAR12*(VAR3-VAR2)
0008 QD=V(I)*VAR32-VAR3*VAR22+VAR1*(VAR22-VAR32)+VAR12*(VAR3-VAR2)
0009 I*VAR12*VAR32-VAR3*VAR22+VAR1*(VAR22-VAR32)+VAR12*(VAR3-VAR2)
0010 D=VAR2*VAR32-VAR3*VAR22+VAR1*(VAR22-VAR32)+VAR12*(VAR3-VAR2)
0011 A=QA/O
0012 B=QB/O
0013 C=QC/O
0014 RETURN
0015 ENO

```



```

0028 A1=A(I)
0029 RH1=RH(I)
0030 A(I)=(A11-A1)/DENOM
0031 RH(I)=(RH11-RH1)/DENOM
0032 A1=A11
0033 RH1=RH11
0034 IF(I-J) 4,5,5
0035 IF(I-J) 4,5,5
0036 C
0037 EQUATE (M+1) THE DIFFERENCE TO ZERO TO DETERMINE H
0038 C WITH H KNOWN, CM81NE THE FUNCTION AND DEVIATION DIFFERENCES
0039 DO 6 I=1,MPLUS2
0040 A(I)=A(I)+RH(I)*H
0041 C COMPUTE POLYNOMIAL COEFFICIENTS
0042 J=M
0043 XJ=RX(J)
0044 I=J
0045 JPLUS1=J+1
0046 DO 9 I,JPLUS1,MPLUS1
0047 A11=A(I)
0048 A11=A11-XJ*A11
0049 A1=A11
0050 I=I-1
0051 J=J-1
0052 IF(J-1) 9,7,7
0053 C
0054 C IF THE REFERENCE DEVIATION IS NOT INCREASING MONOTONICALLY
0055 THEN EXIT
0056 HMAX=ABS(H)
0057 A11=MAX(0,PREVH) GO TO 29
0058 C RETURN THE INDEX, IMAX, AND VALUE, HMAX, OF THE LARGEST ABSOLUTE
0059 ERROR FOR ALL SAMPLE POINTS
0060 A(MPLUS2)=HMAX
0061 PREVH=HMAX
0062 IMAX=X(I)
0063 HMAX=H
0064 J=J-1
0065 DO 11 I=1,N
0066 IF(I.EQ.RJ) GO TO 11
0067 X1=X(I)
0068 X1=X1*(MPLUS1)
0069 K=K+1
0070 H1=H1+X1+A(K)
0071 IF(K-1) 12,12,12
0072 H1=H1
0073 ABSH1=ABS(H1)
0074 IF(ABSH1.GE.HMAX) GO TO 11
0075 HMAX=ABSH1
0076 HMAX=H1
0077 GO TO 10
0078 IF(J.GE.MPLUS2) GO TO 10
0079 J=J+1
0080 C
0081 C
0082 C
0083 C IF THE MAXIMUM ERROR OCCURS AT A NONREFERENCE POINT, EXCHANGE THIS
0084 POINT WITH THE NEAREST REFERENCE POINT HAVING AN ERROR OF THE
0085 SAME SIGN AND REPEAT
0086 IF( MAX.EQ.R(I)) RETURN
0087 DO 14 I=2,MPLUS2
0088 IF( IMAX.LT.R(I)) GO TO 15
0089 I=I+1
0090 NEXT I=H
0091 IF( I-1/2*21.NE.0) NEXT I=-H
0092 IF( IMAX*NEXT I.GE.0) GO TO 115

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FORTRAN IV G LEVEL 18      CHBFT      GO TO 116
0091      IF(IMAX.GE.R(1))
0092      J1=MPLUS2
0093      117 R(J1)=R(J)
0094      J1=J
0095      IF(J1.118,117,117
0096      GO TO 2
0097      118 R(J1)=IMAX
0098      GO TO 2
0099      116 IF(IMAX.LE.R(MPLUS2)) GO TO 120
0100      J=1
0101      DO 121 J1=1,MPLUS2
0102      R(J1)=R(J1)
0103      121 J=J1
0104      R(MPLUS2)=IMAX
0105      GO TO 2
0106      115 R(J1)=IMAX
0107      GO TO 2
0108      120 R(J1)=IMAX
0109      GO TO 2
0110      END
0111
```

```

0001      FUNCTION YC(XBAR,C,M)      YC
0002      DIMENSION C(1)
0003      YC=0BAR.EQ.0.} YC=C(1)
0004      IF(XBAR.EQ.0.} GOTO 10
0005      M1=M+1
0006      DO 1 I=1,M1
0007      1 YC=YC+C(I)*XBAR**(I-1)
0008      10 CONTINUE
0009      1000 RETURN
0010      END
0011

```

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13. ABSTRACT <p>The report presents a computer program for prediction of performance of single-stage axial turbines of given geometry. The three-dimensional method developed by Vavra is applied, taking account of streamline curvatures and slopes, as well as enthalpy and entropy gradients in the solutions of the equation of motion, and of boundary layer thicknesses in the continuity equation.</p> <p>A choice among five different loss correlation methods and two flow angles correlations is offered. Loss coefficients and flow angles are automatically calculated from blading geometry and actual flow conditions for every streamline, according to the selected correlation method.</p> <p>A fair agreement of predictions with several actual turbines experimental results was found in ref. [4], where also the applicability of different available correlations is discussed.</p>

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KEY WORDS

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LINK B

LINK C

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